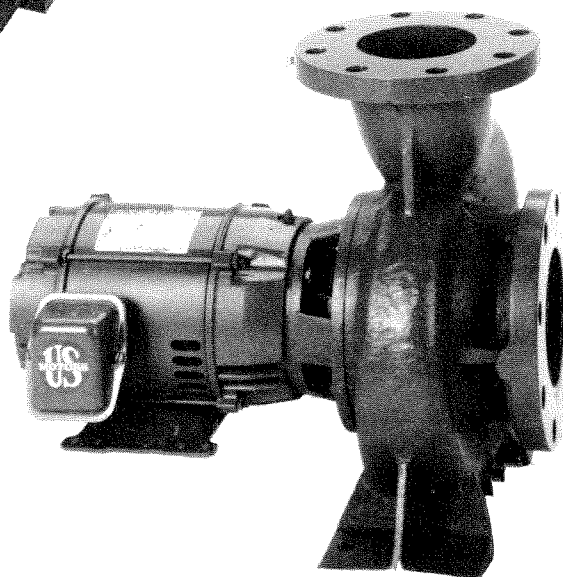
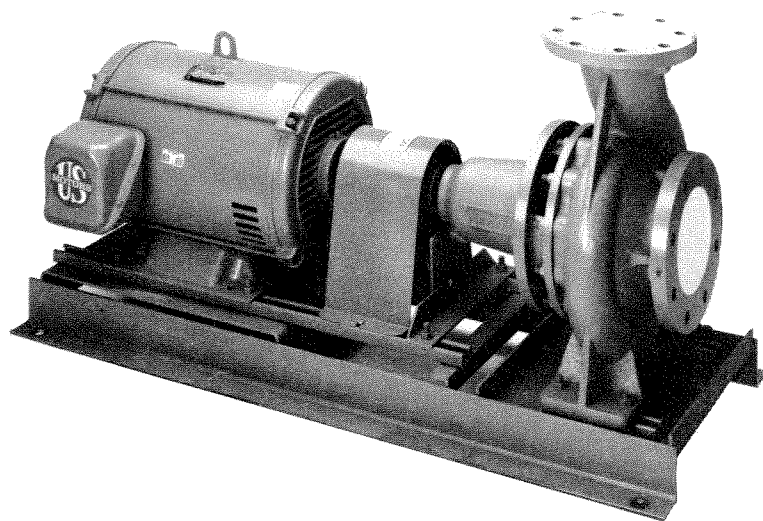




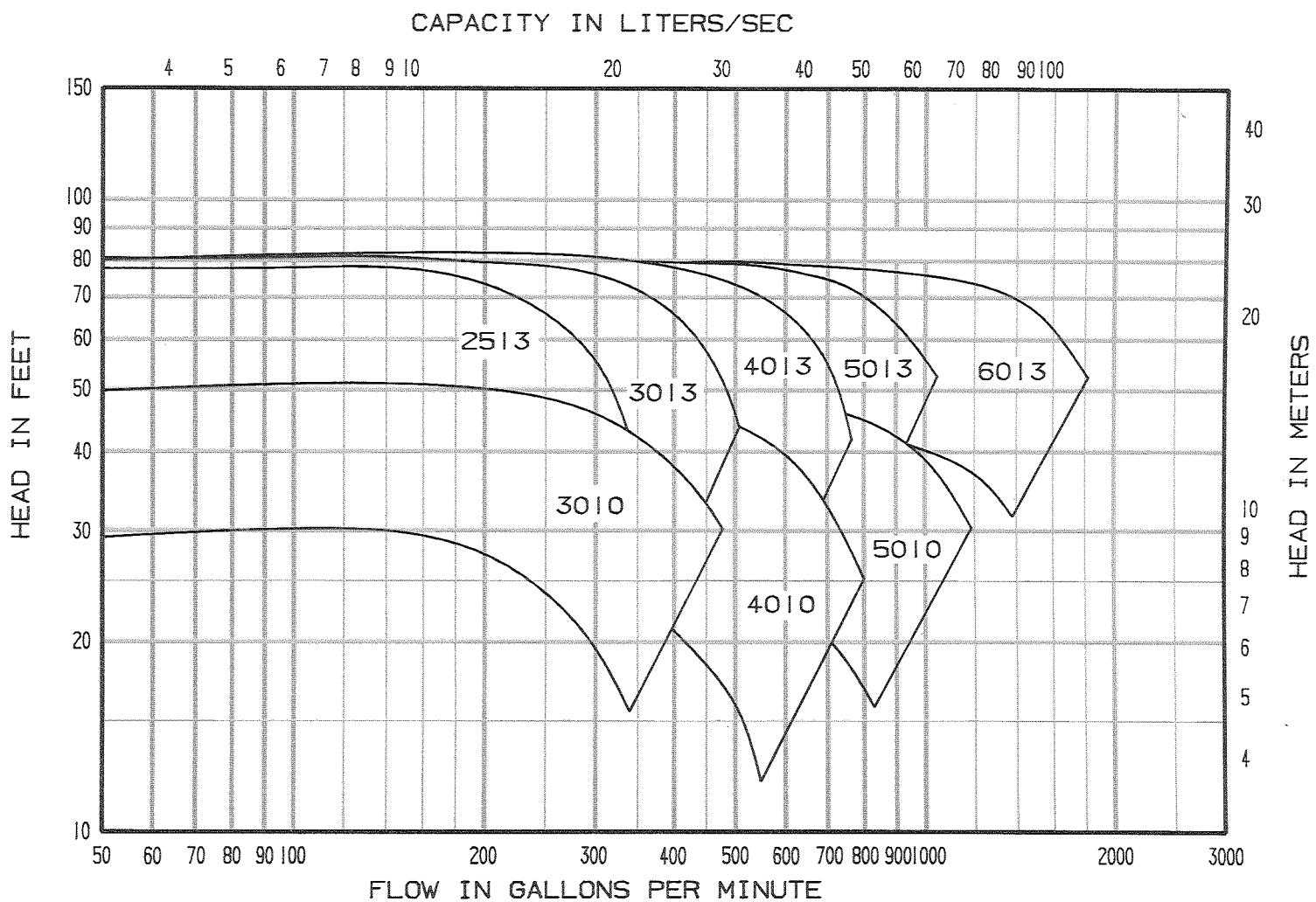
COMPARE. YOU'LL TAKE TACO.

PERFORMANCE CURVES

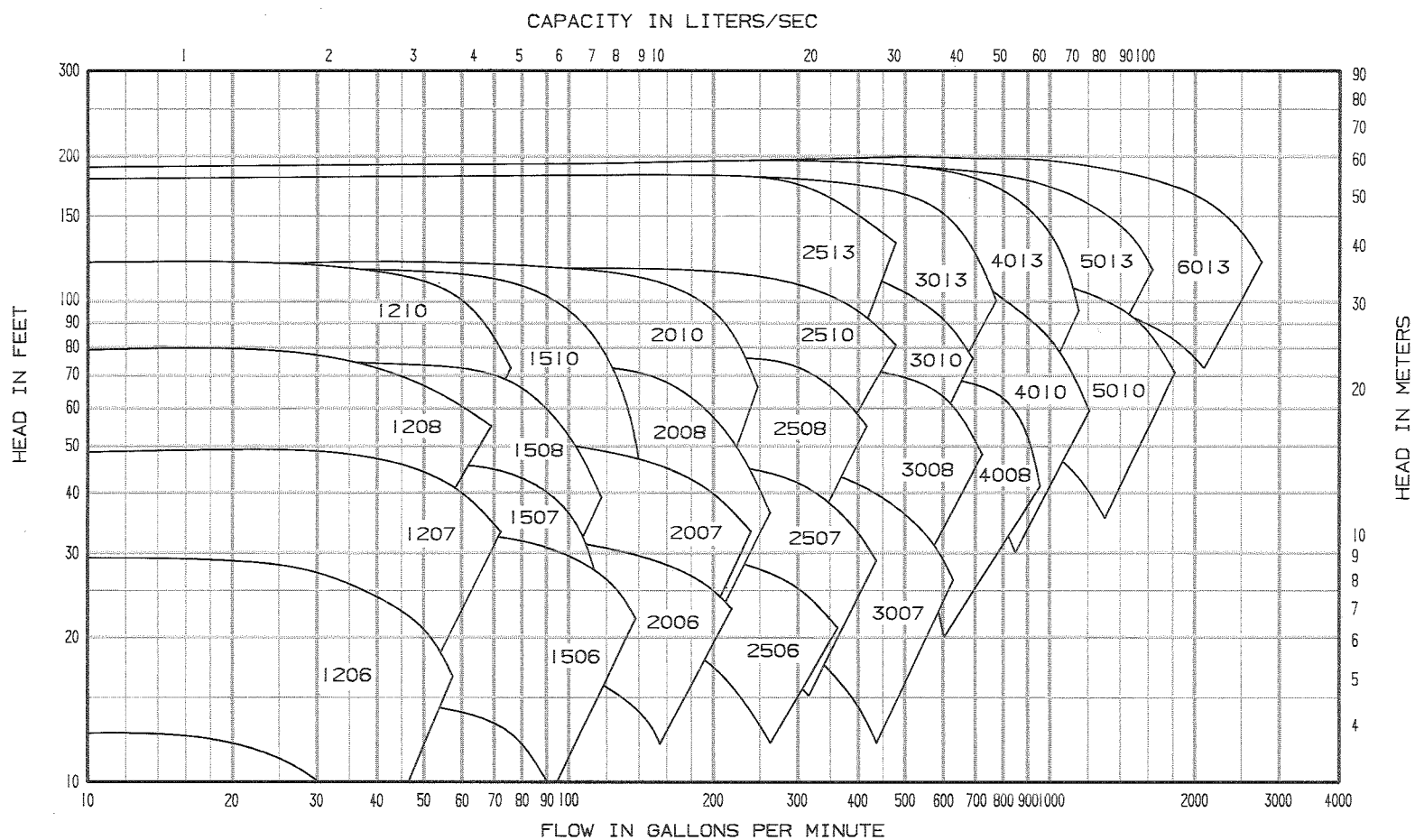


"CE" Close-Coupled End Suction Pump

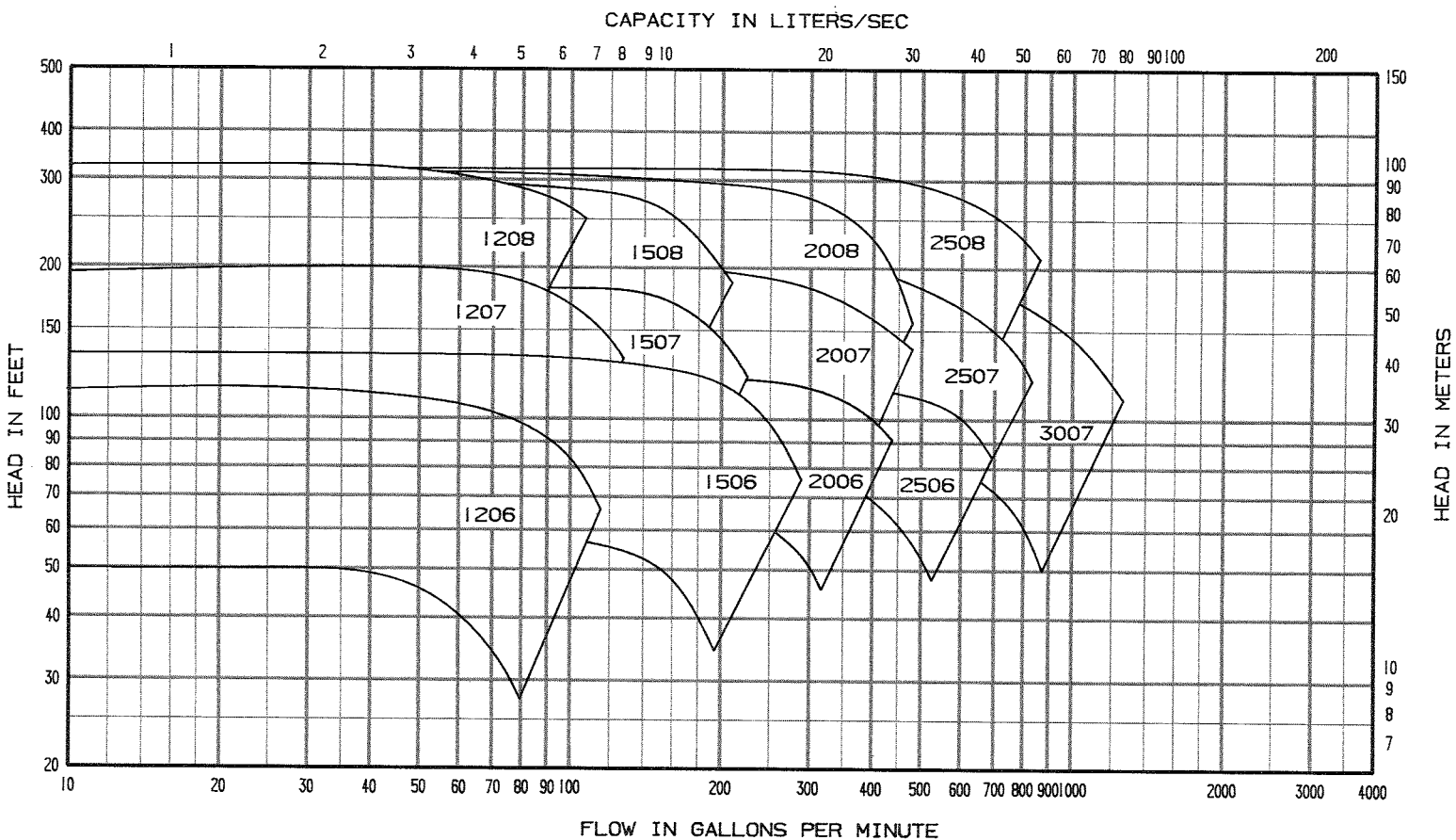
"FE" Frame Mounted End Suction Pump



**Close-Coupled End Suction Pump
Frame Mounted End Suction Pump
Quick Selection Curve
1160 RPM**



**Close-Coupled End Suction Pump
Frame Mounted End Suction Pump
Quick Selection Curve
1760 RPM**



Close-Coupled End Suction Pump
Frame Mounted End Suction Pump
Quick Selection Curve
3500 RPM

CORRECTION FACTORS
FOR OTHER SPEEDS

For speeds other than 1760, hydraulic performance and B.H.P. requirements must be corrected before a selection can be made. The following multipliers are used to correct performance and B.H.P.

G.P.M.	Head	B.H.P.
1160 to 1760	1.52	2.31
1760 to 1160		.28
1460 to 1760	1.21	1.46
1760 to 1460		.58
2900 to 3500	1.19	1.42
3500 to 2900		.58

Sizing Procedure

1. Using the proper multipliers correct G.P.M. and head.
2. Make pump selection as usual using corrected performance.
3. Determine max. B.H.P. requirement for pump selected.
4. Correct B.H.P. using multiplier, to lower speed.
5. Select motor based on Step 4. Use service factor if applicable.

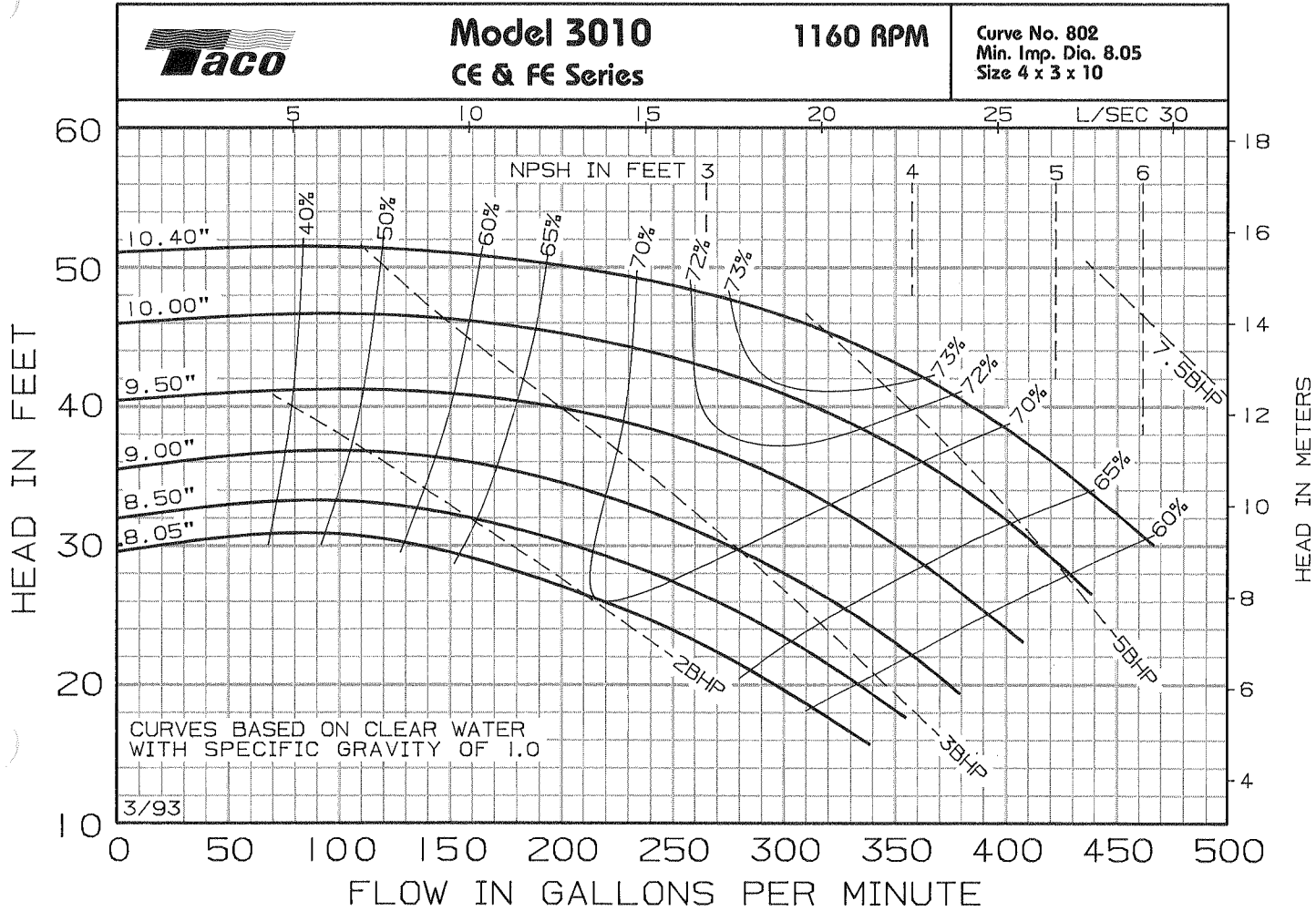
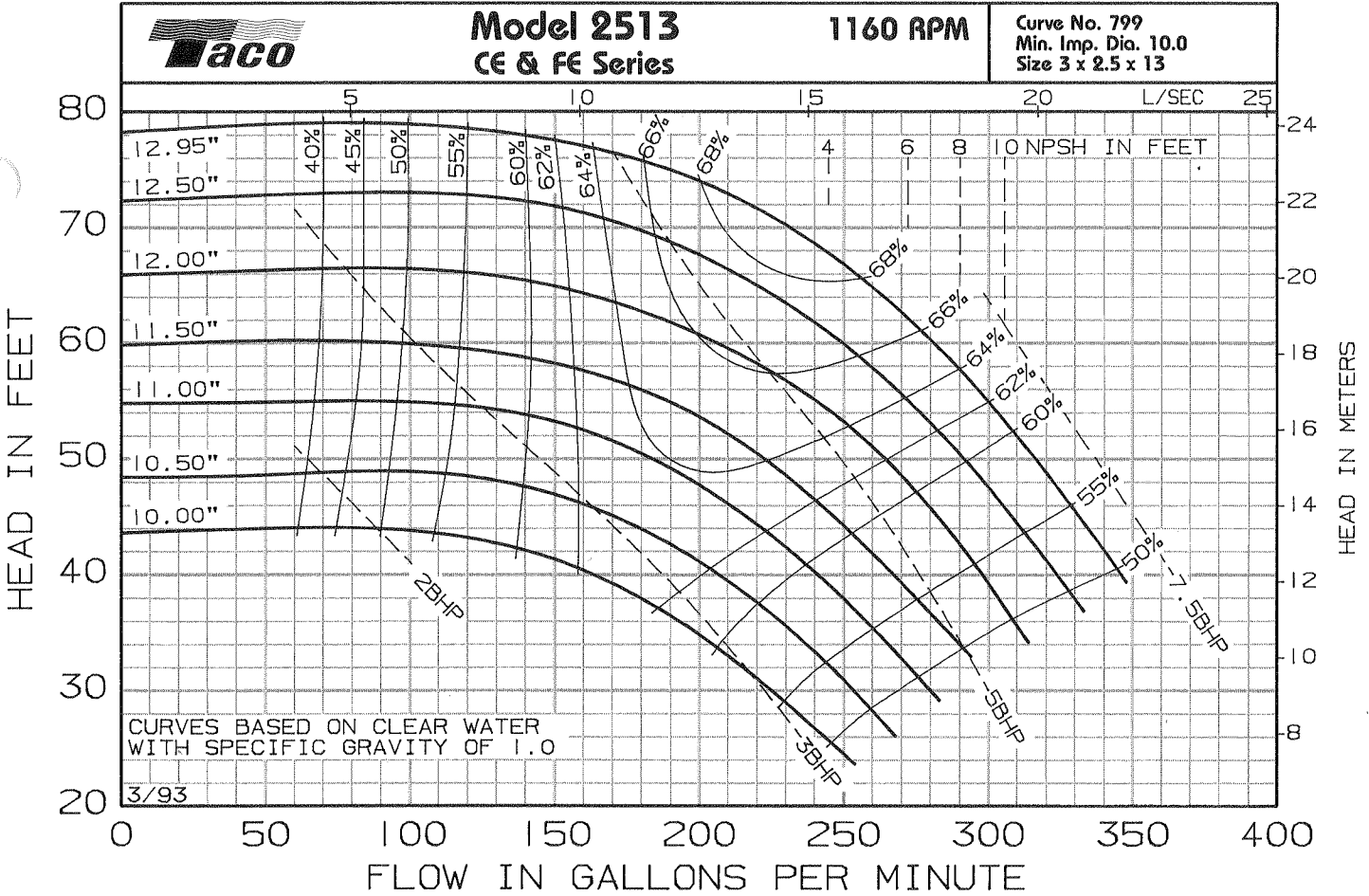
PUMP FORMULAS

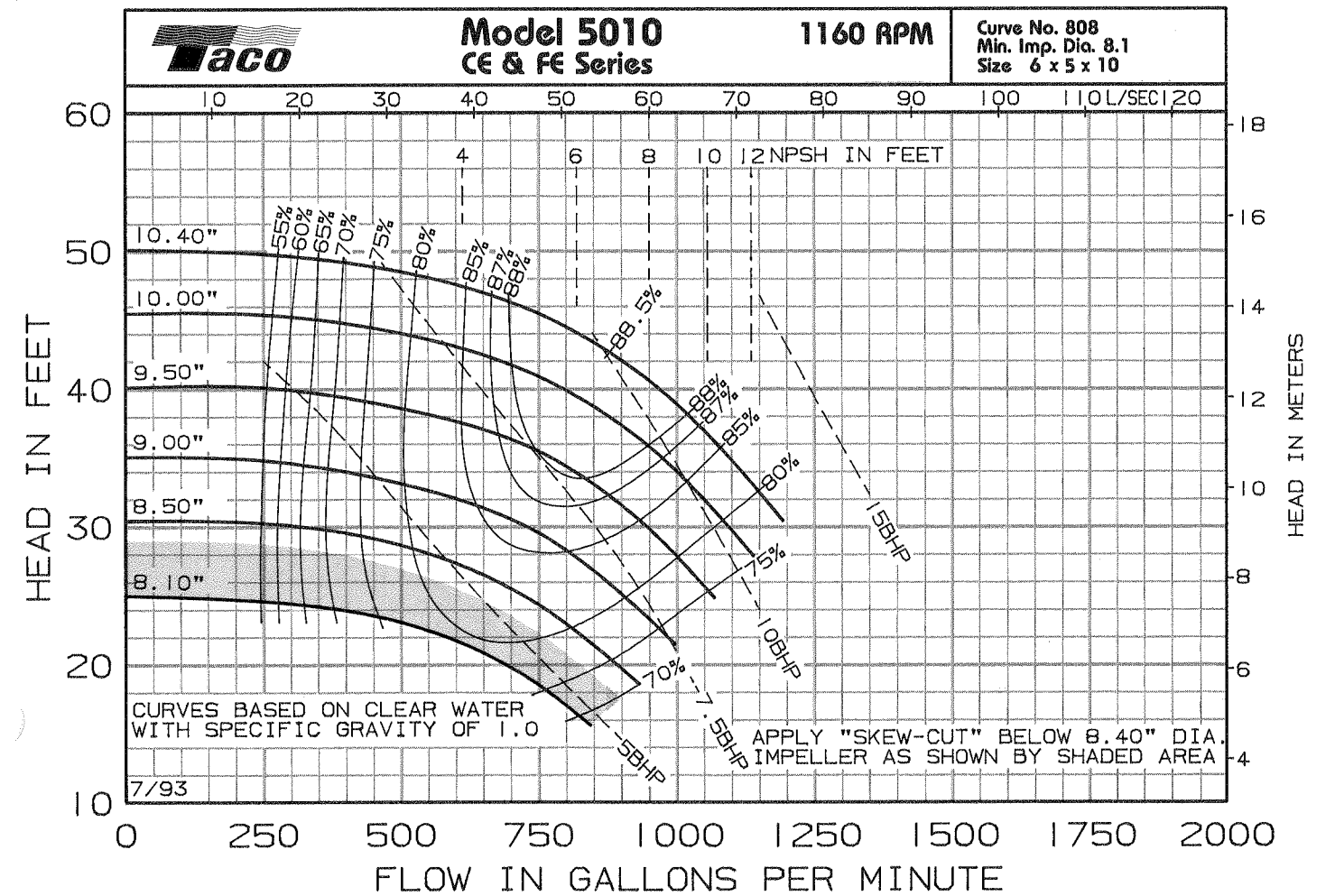
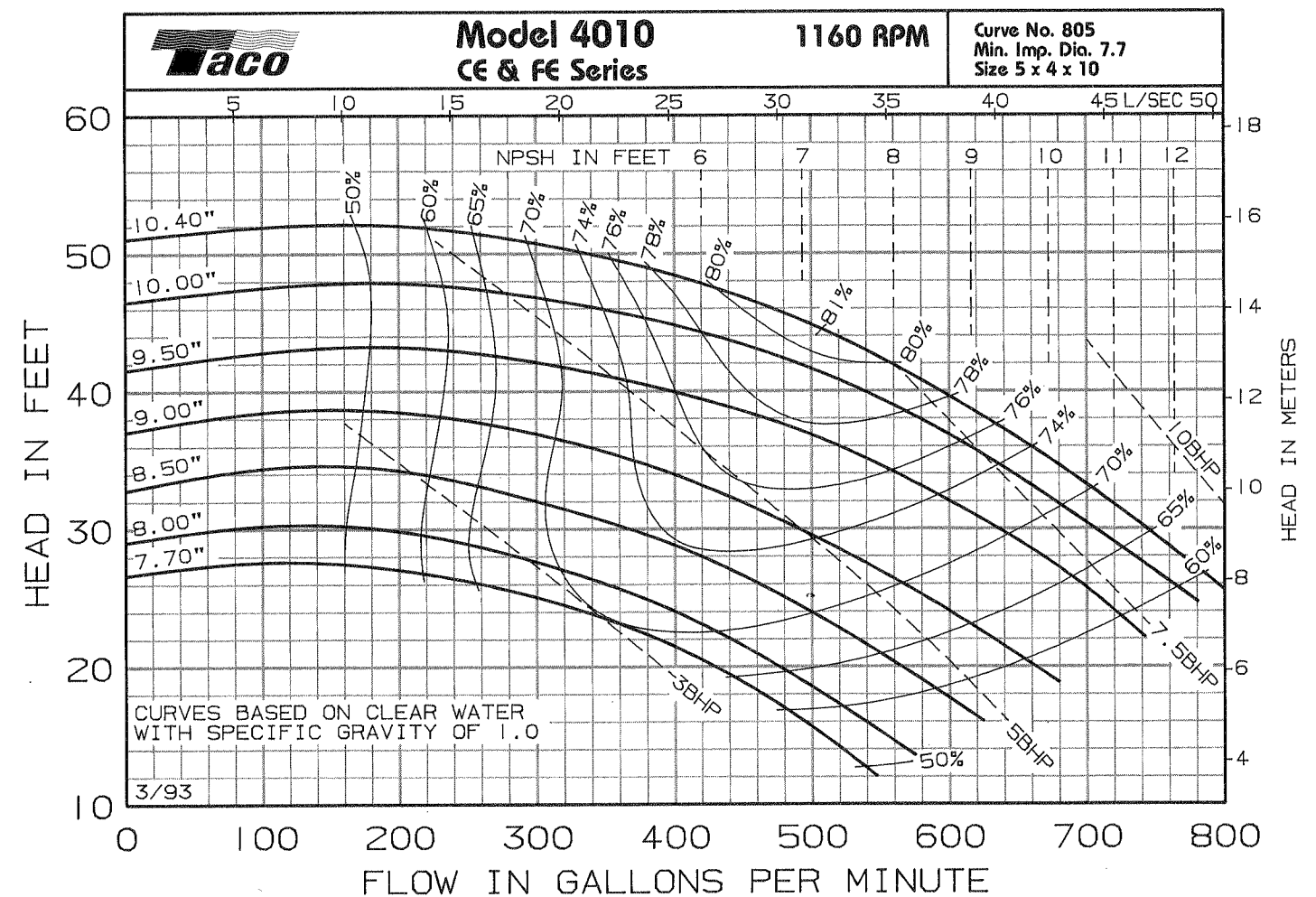
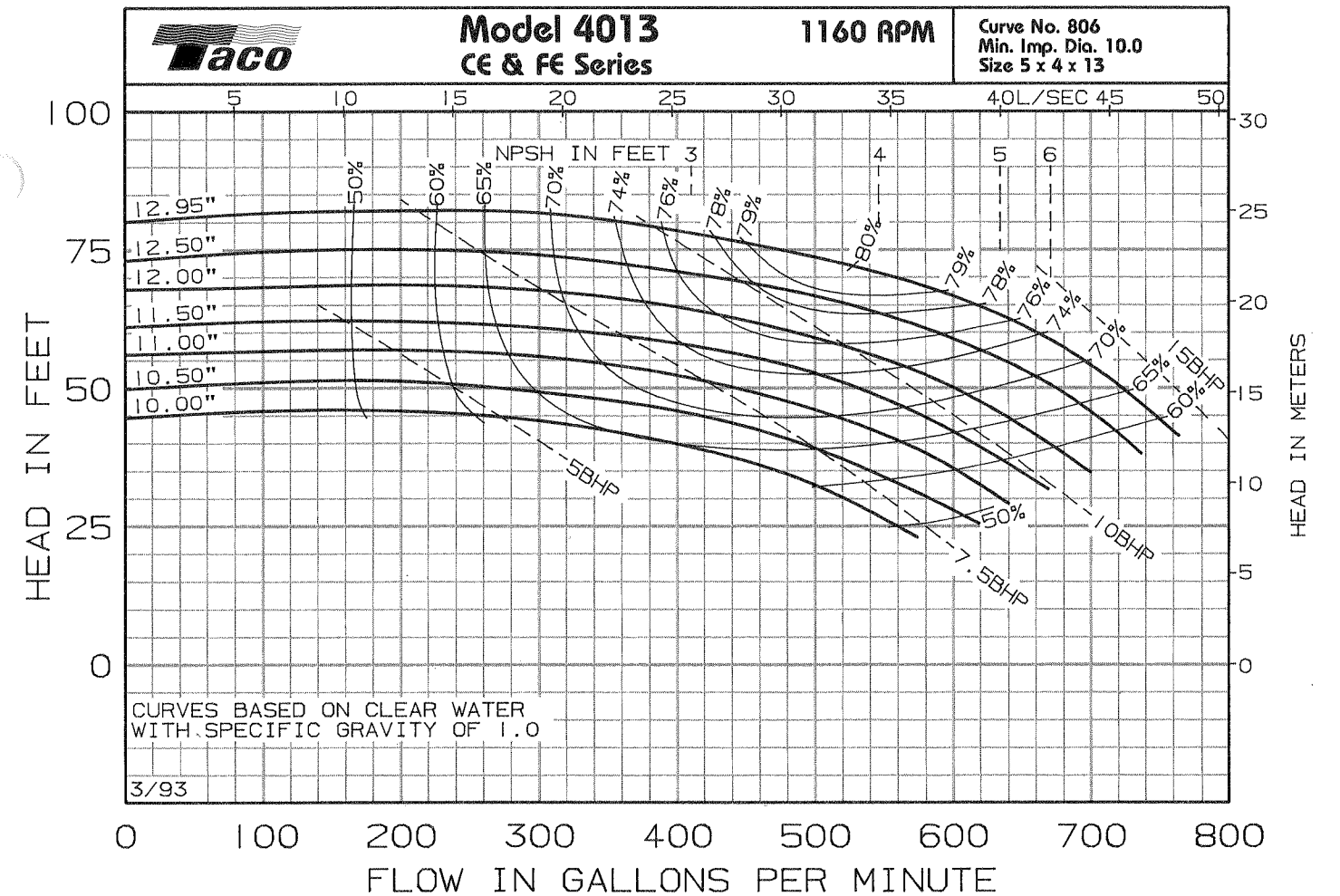
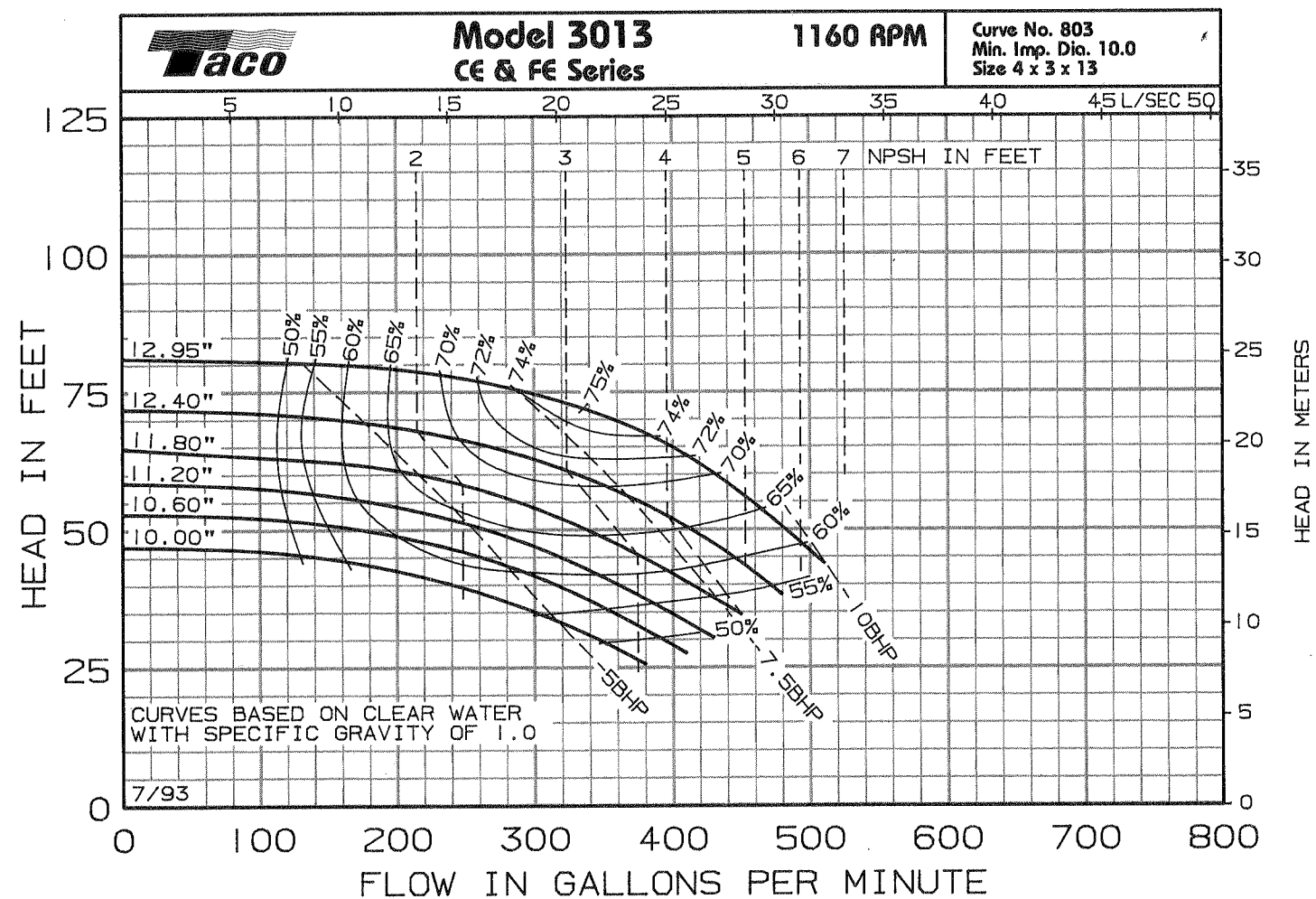
Pressure (PSI)	=	$\frac{\text{Head (Feet)} \times \text{Specific Gravity}}{2.31}$
Head (Feet)	=	$\frac{\text{Pressure (PSI)} \times 2.31}{\text{Specific Gravity}}$
Vacuum (Inches of Mercury)	=	$\text{Dynamic Suction Lift (Feet)} \times .883 \times \text{Specific Gravity}$
Horsepower (Brake)	=	$\frac{\text{GPM} \times \text{Head (Feet)} \times \text{Specific Gravity}}{3960 \times \text{Pump Efficiency}}$
Horsepower (Water)	=	$\frac{\text{GPM} \times \text{Head (Feet)} \times \text{Specific Gravity}}{3960}$
Efficiency (Pump)	=	$\frac{\text{Horsepower (Water)} \times 100 \text{ Per Cent}}{\text{Horsepower (Brake)}}$
NPSH (Available)	=	Positive Factors — Negative Factors

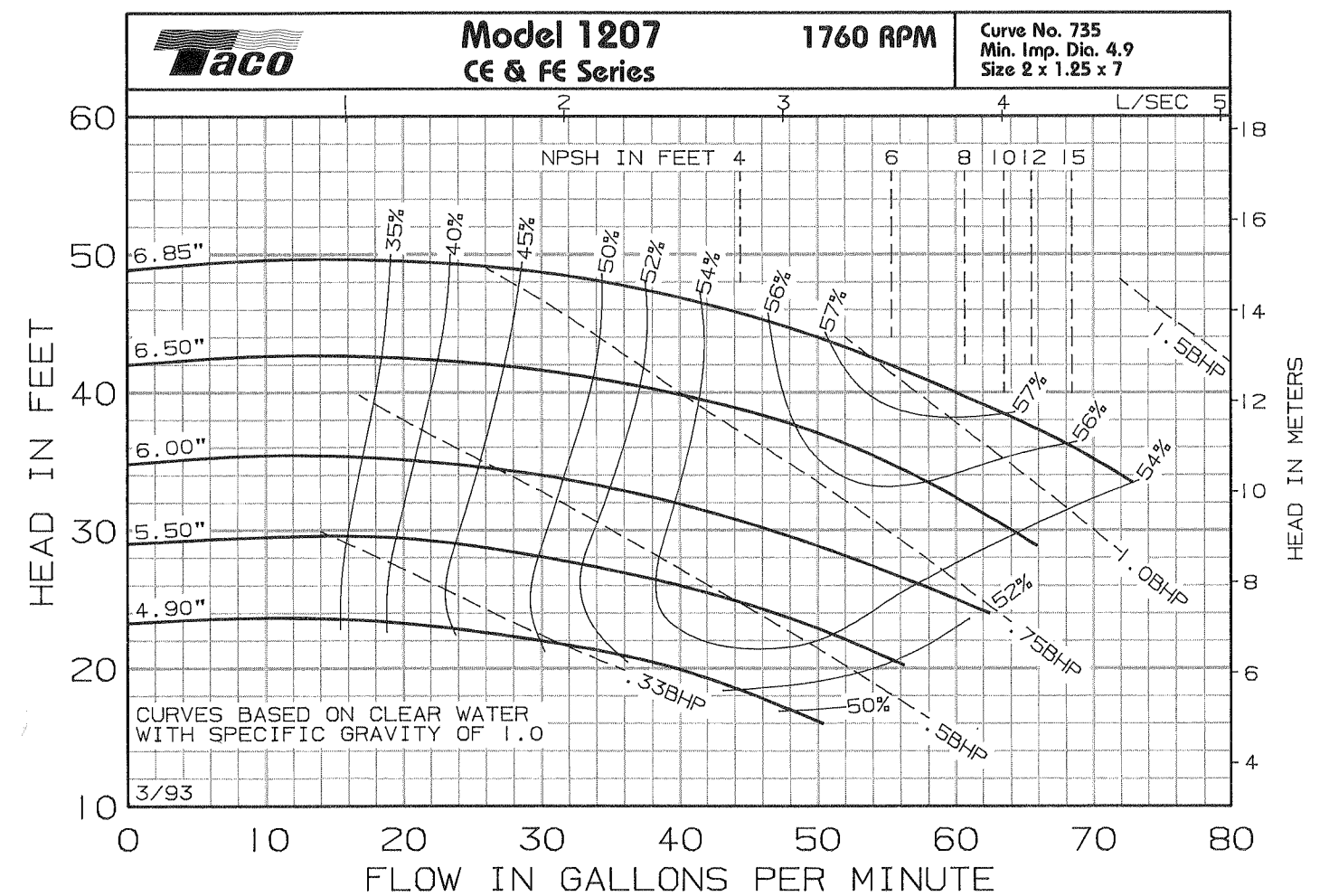
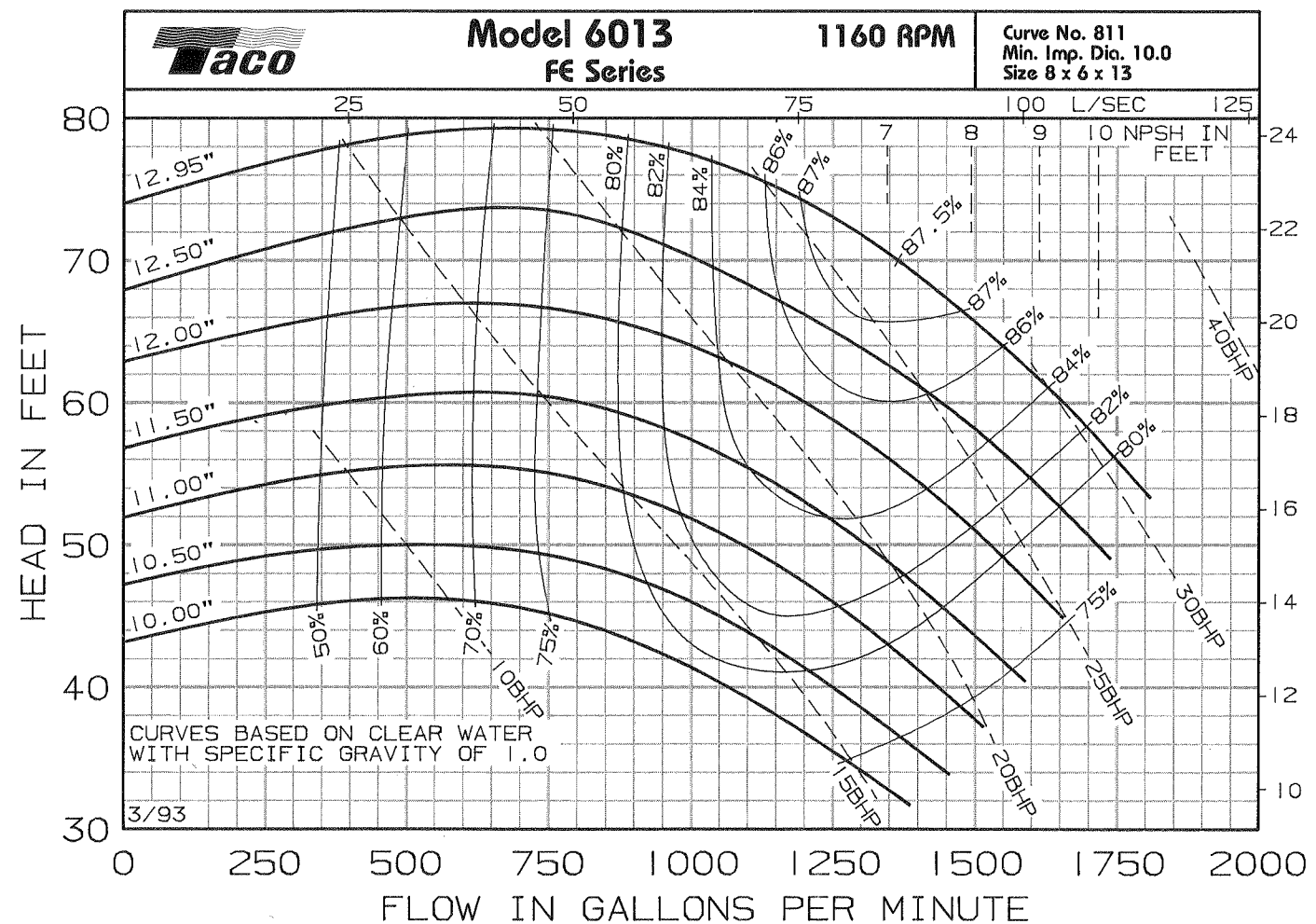
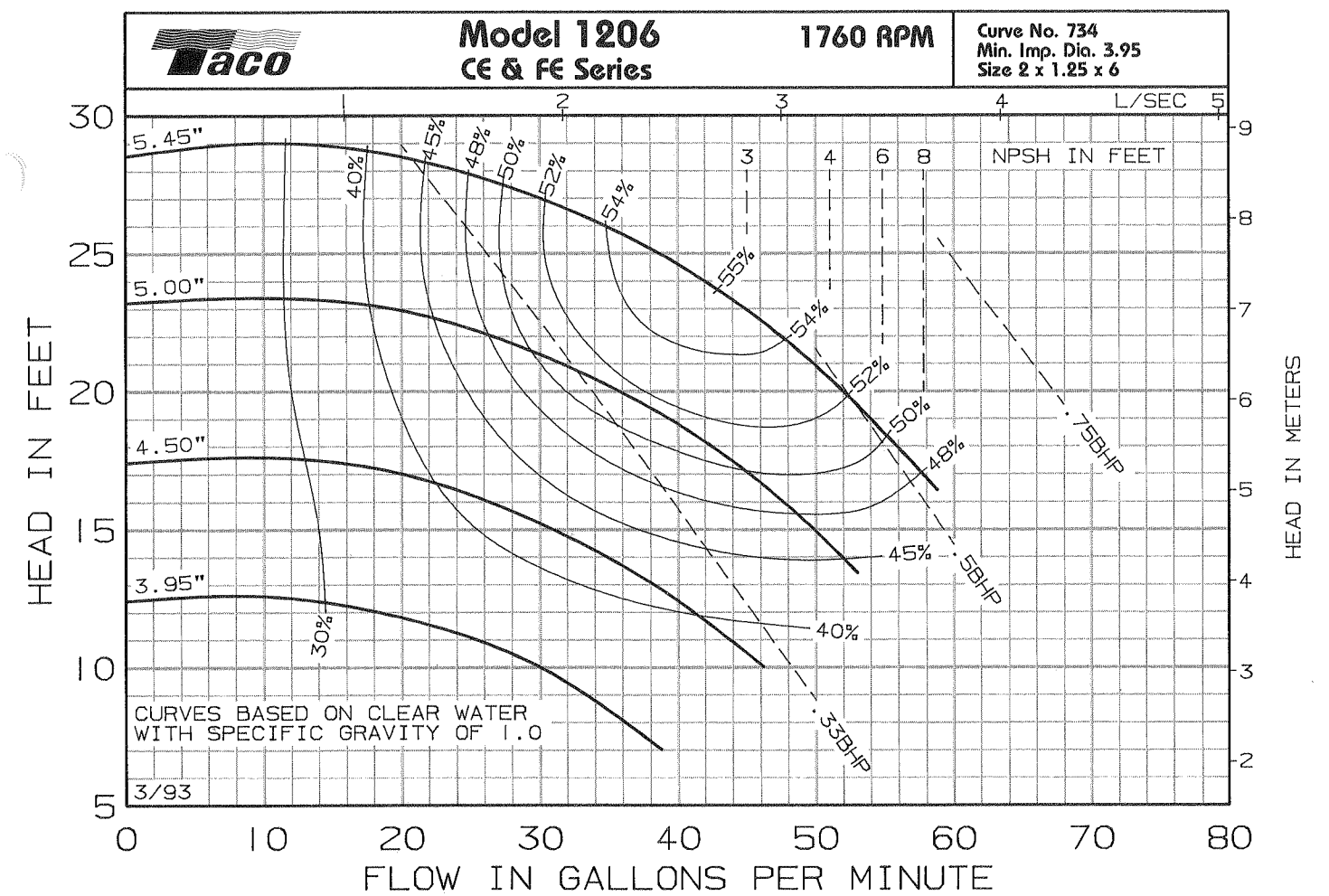
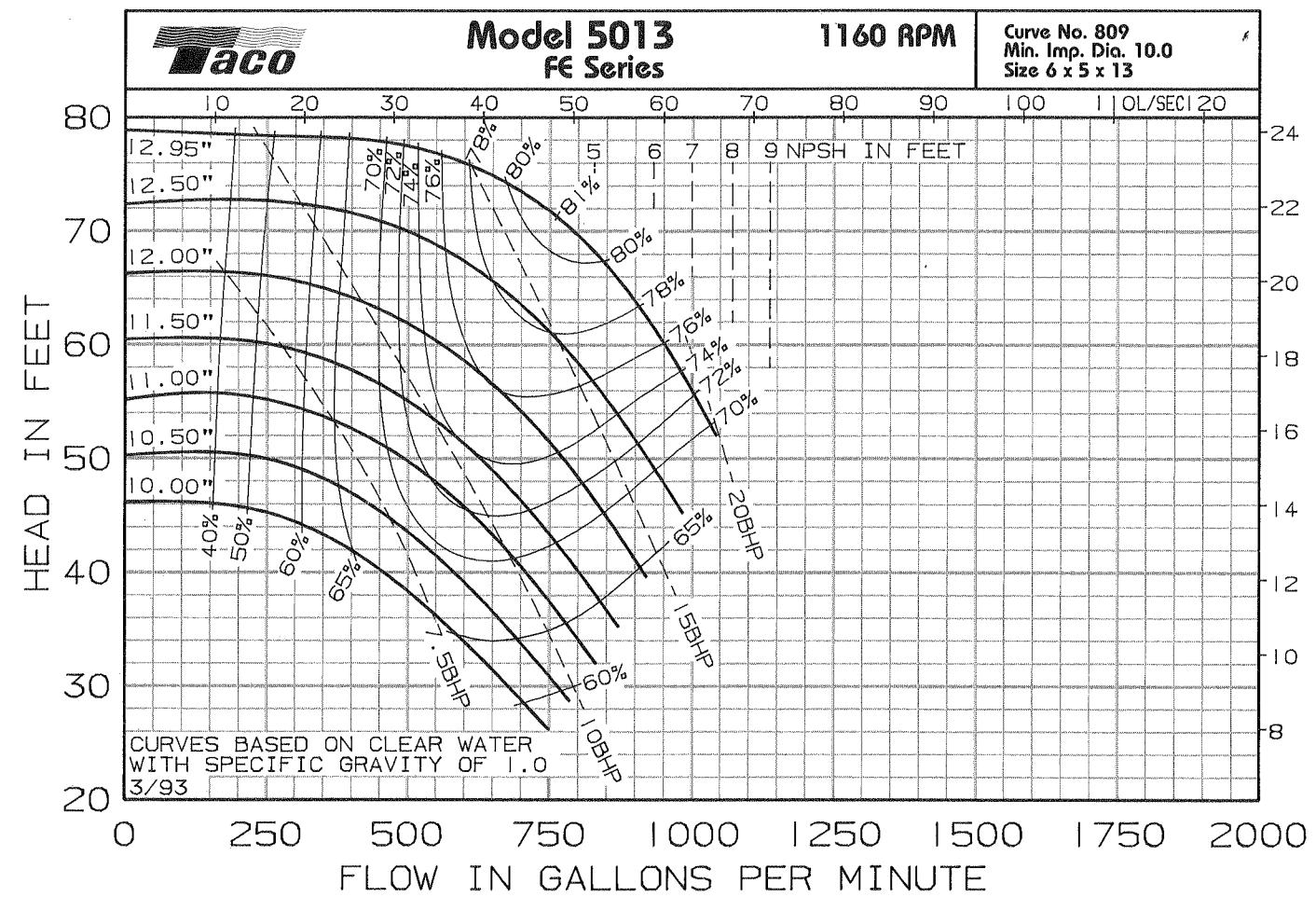
Affinity Laws: Effect of change of speed or impeller diameter on centrifugal pumps.

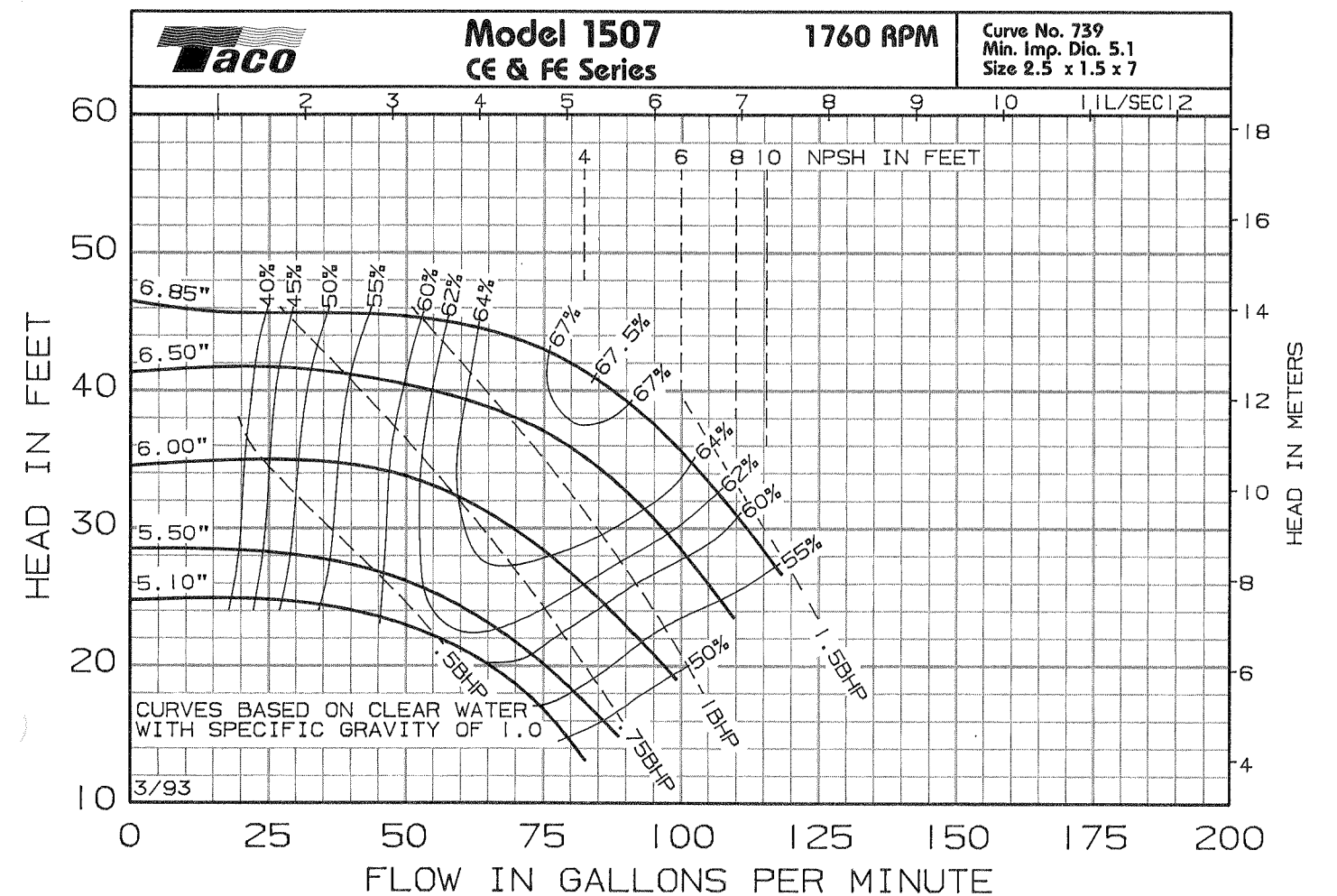
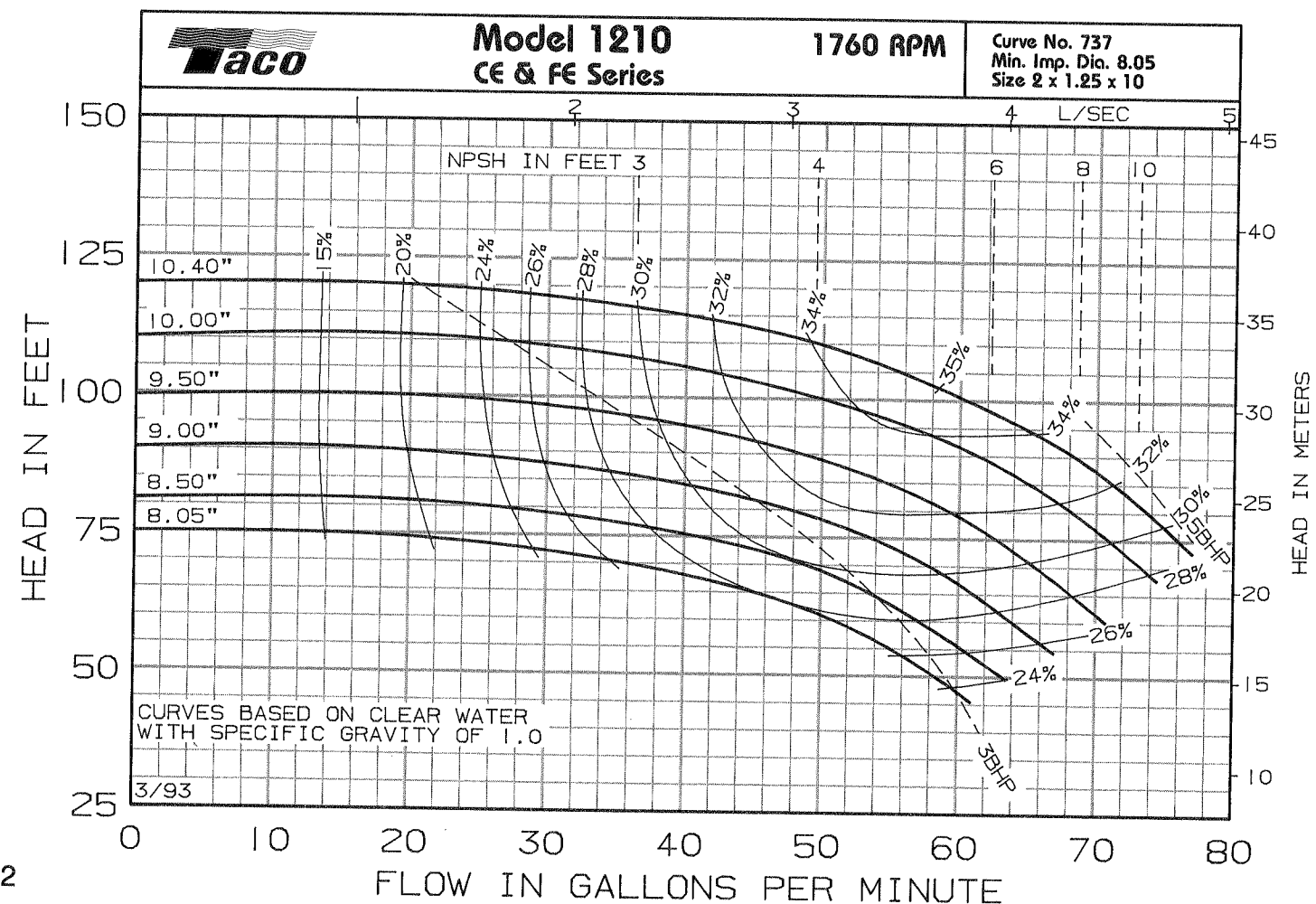
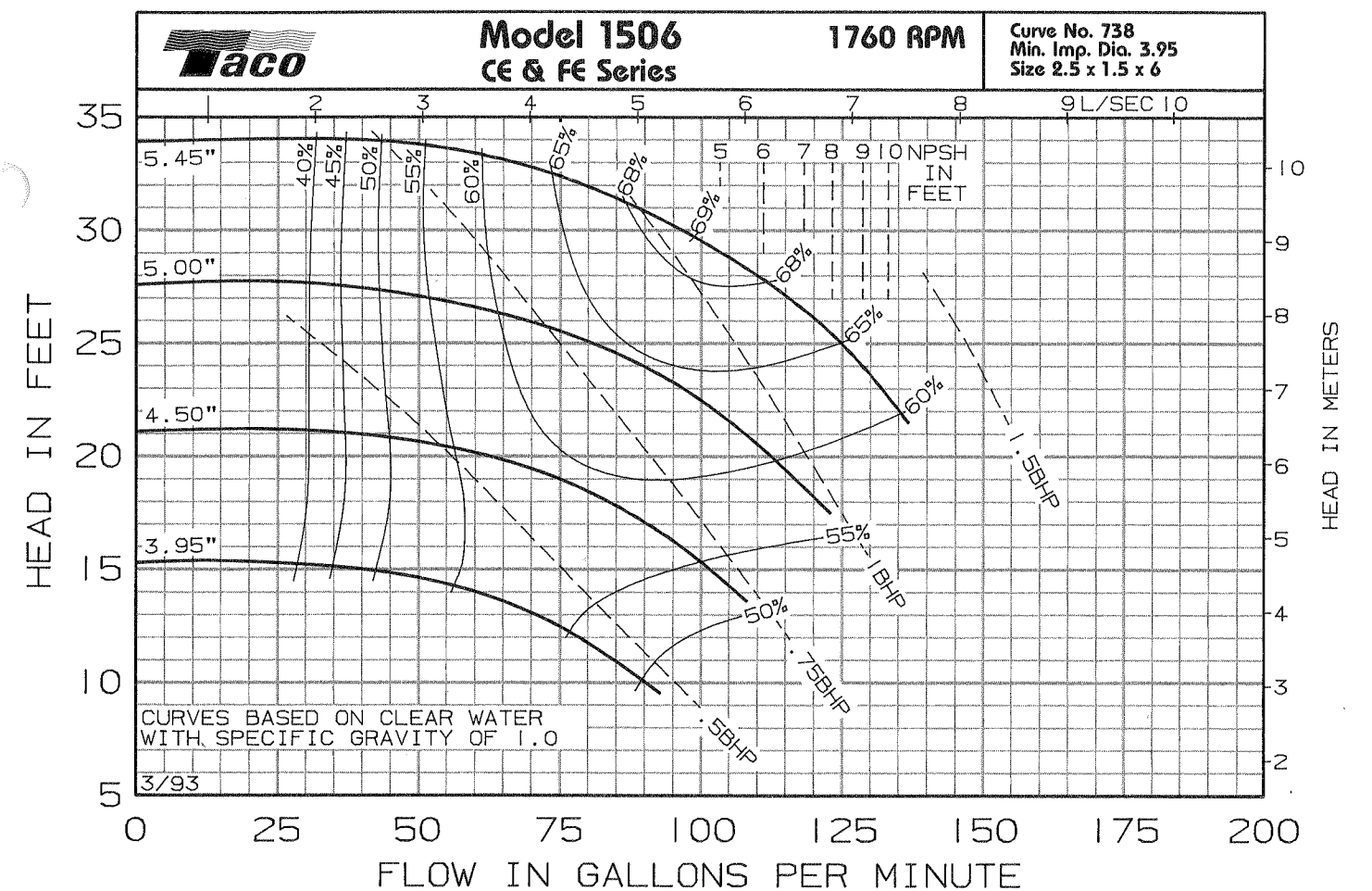
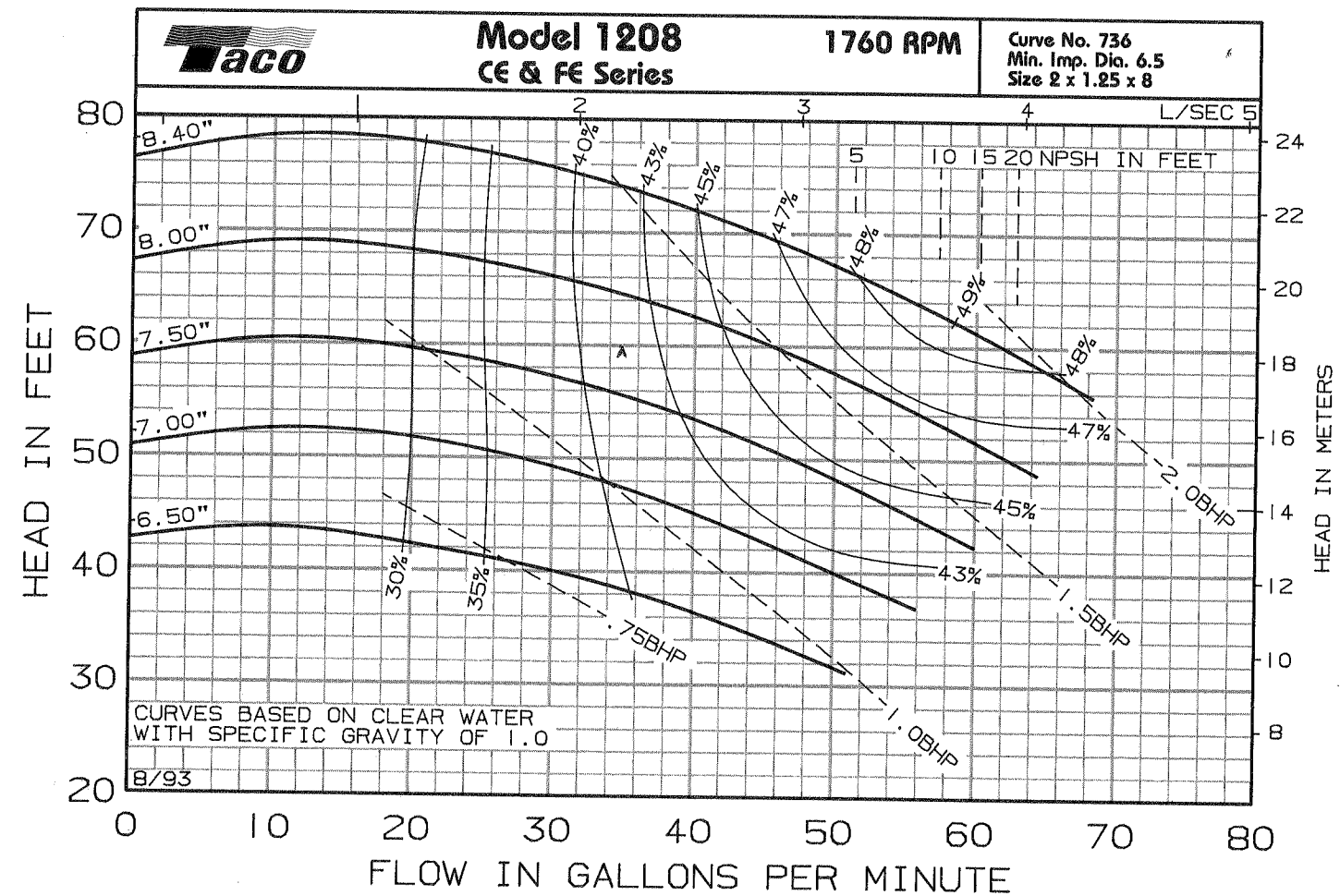
	GPM Capacity	Ft. Head	BHP
Impeller Diameter Change	$Q_2 = \frac{D_2}{D_1} Q_1$	$H_2 = \left(\frac{D_2}{D_1}\right)^2 H_1$	$P_2 = \left(\frac{D_2}{D_1}\right)^3 P_1$
Speed Change	$Q_2 = \frac{\text{RPM}_2}{\text{RPM}_1} Q_1$	$H_2 = \left(\frac{\text{RPM}_2}{\text{RPM}_1}\right)^2 H_1$	$P_2 = \left(\frac{\text{RPM}_2}{\text{RPM}_1}\right)^3 P_1$

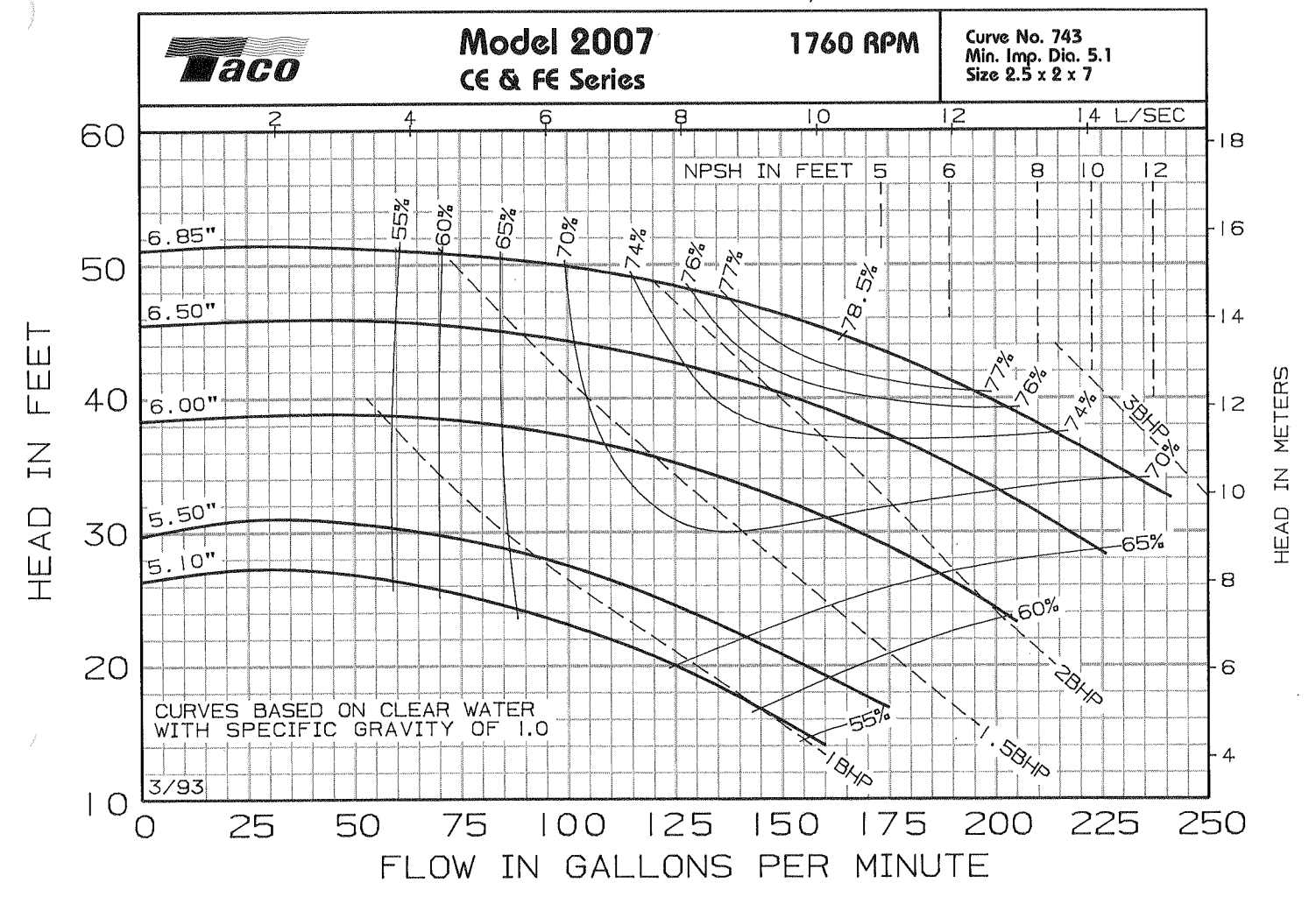
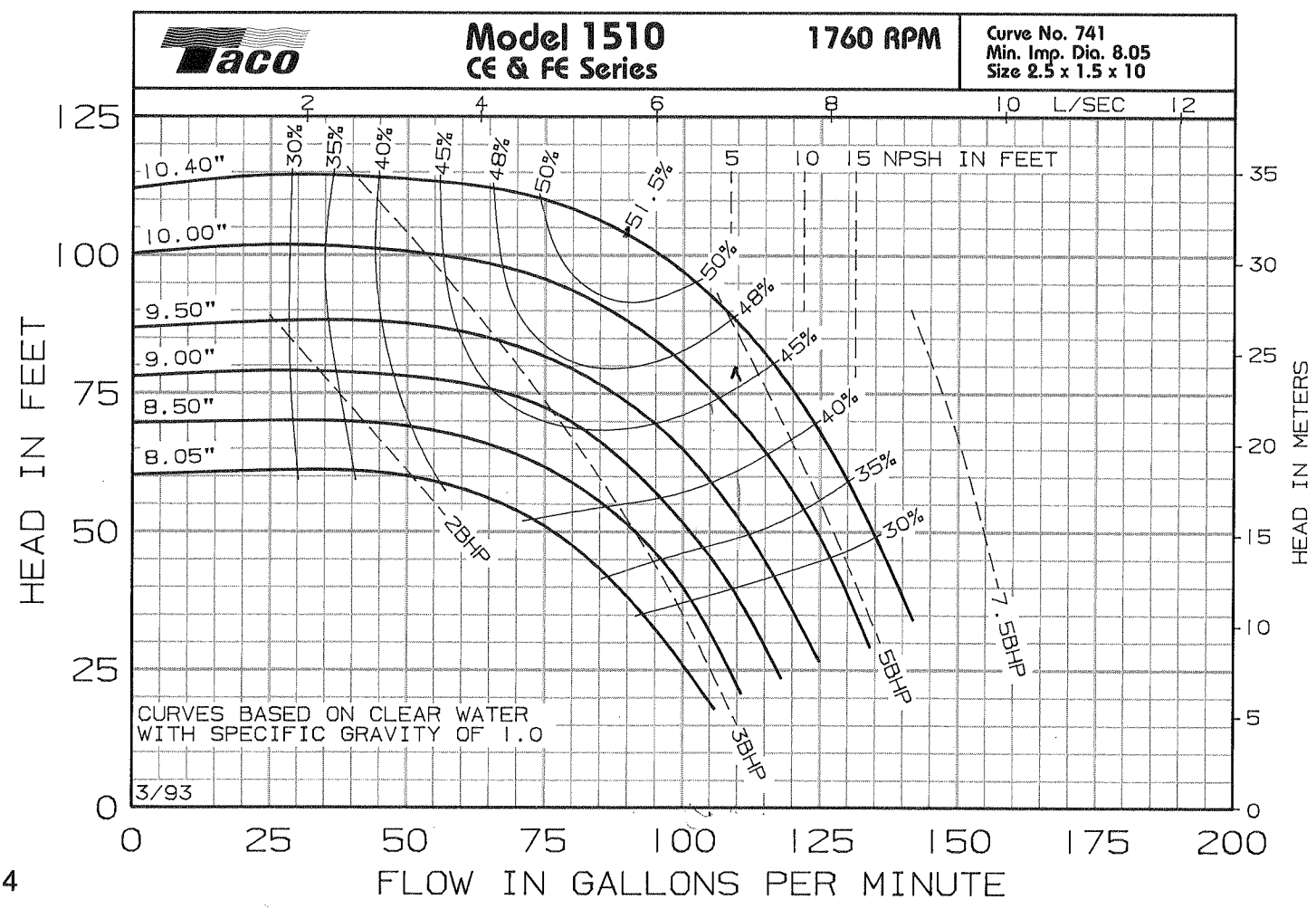
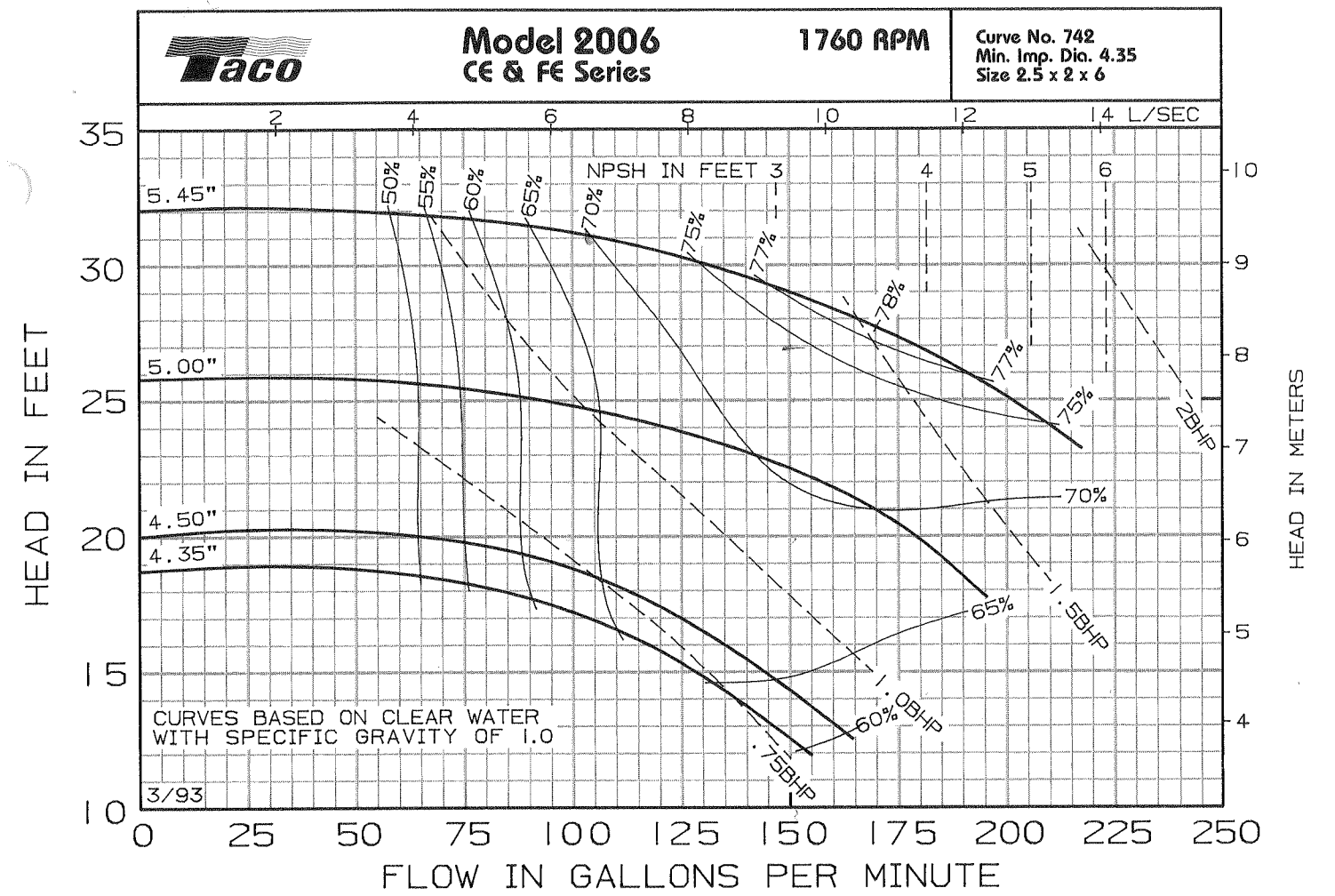
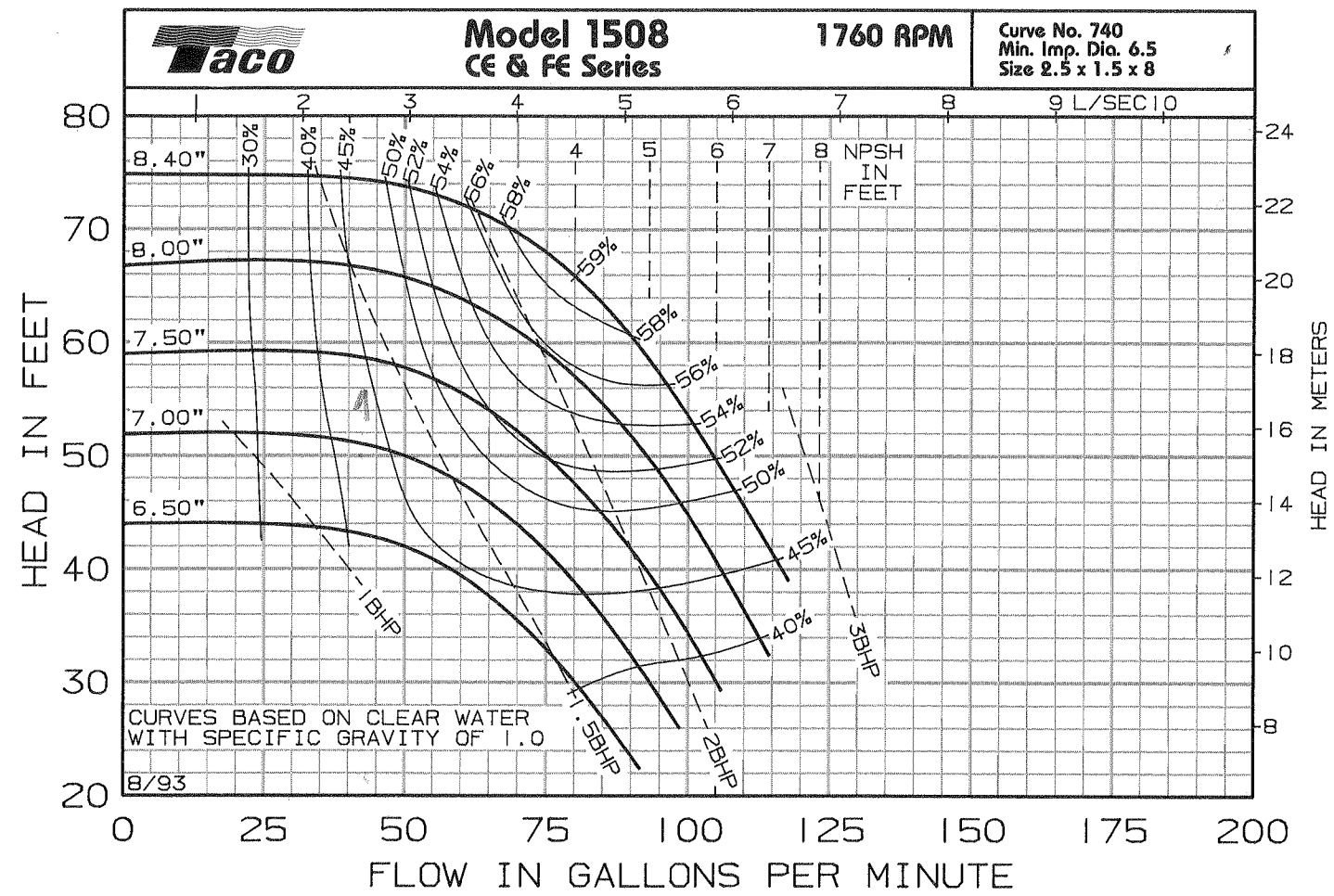
Where Q = GPM, H = Head, P = BHP, D = Impeller Dia., RPM = Pump Speed

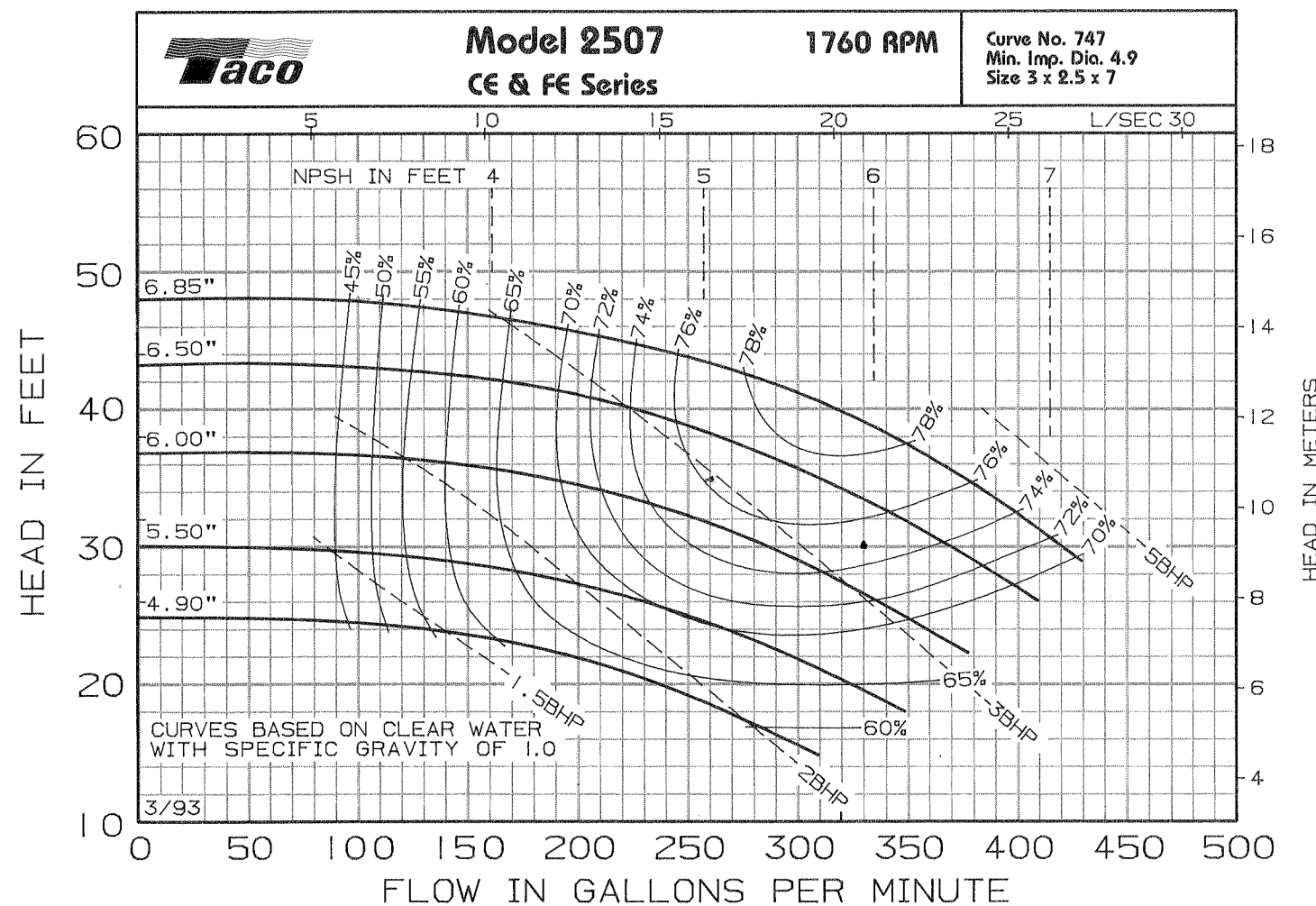
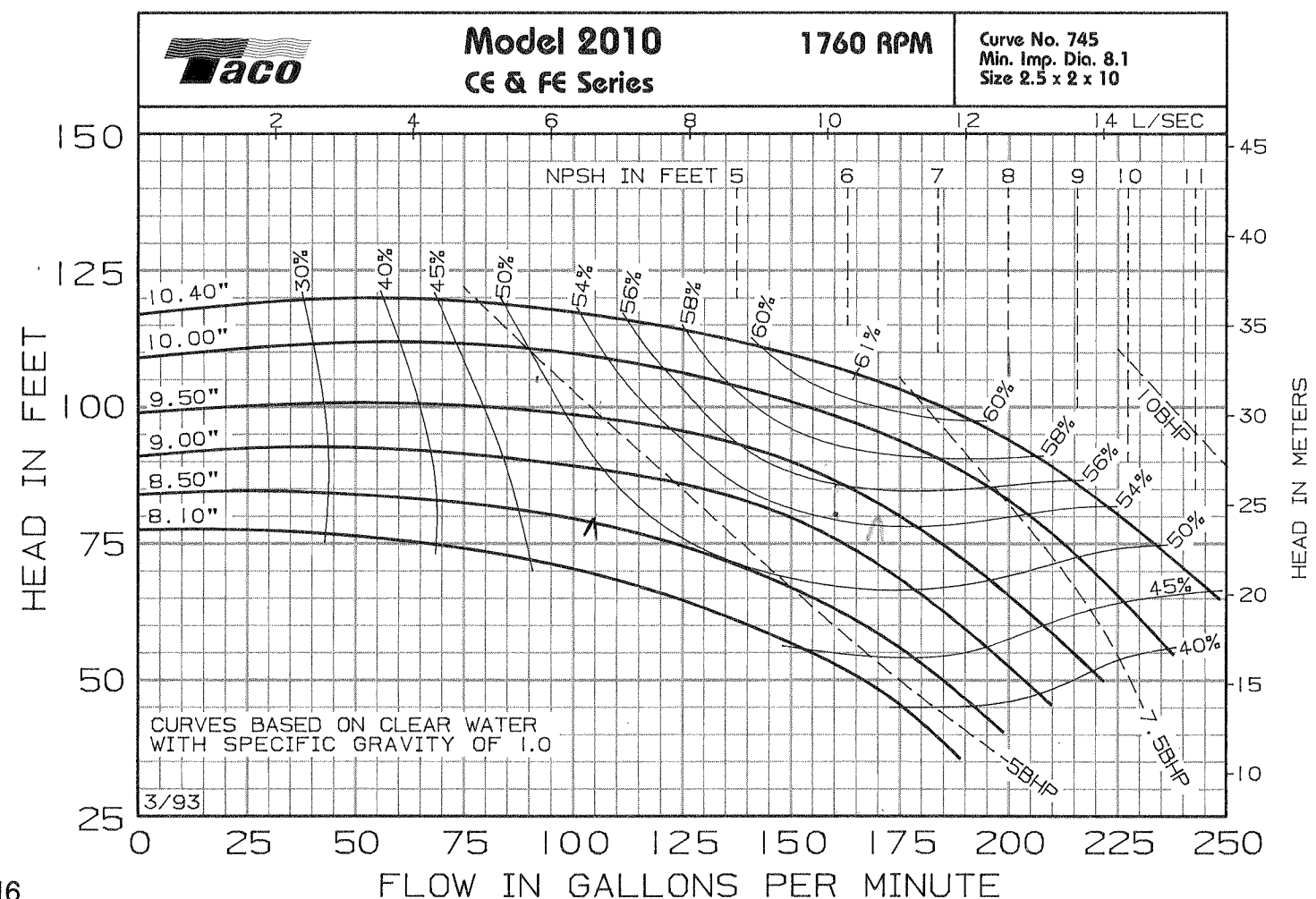
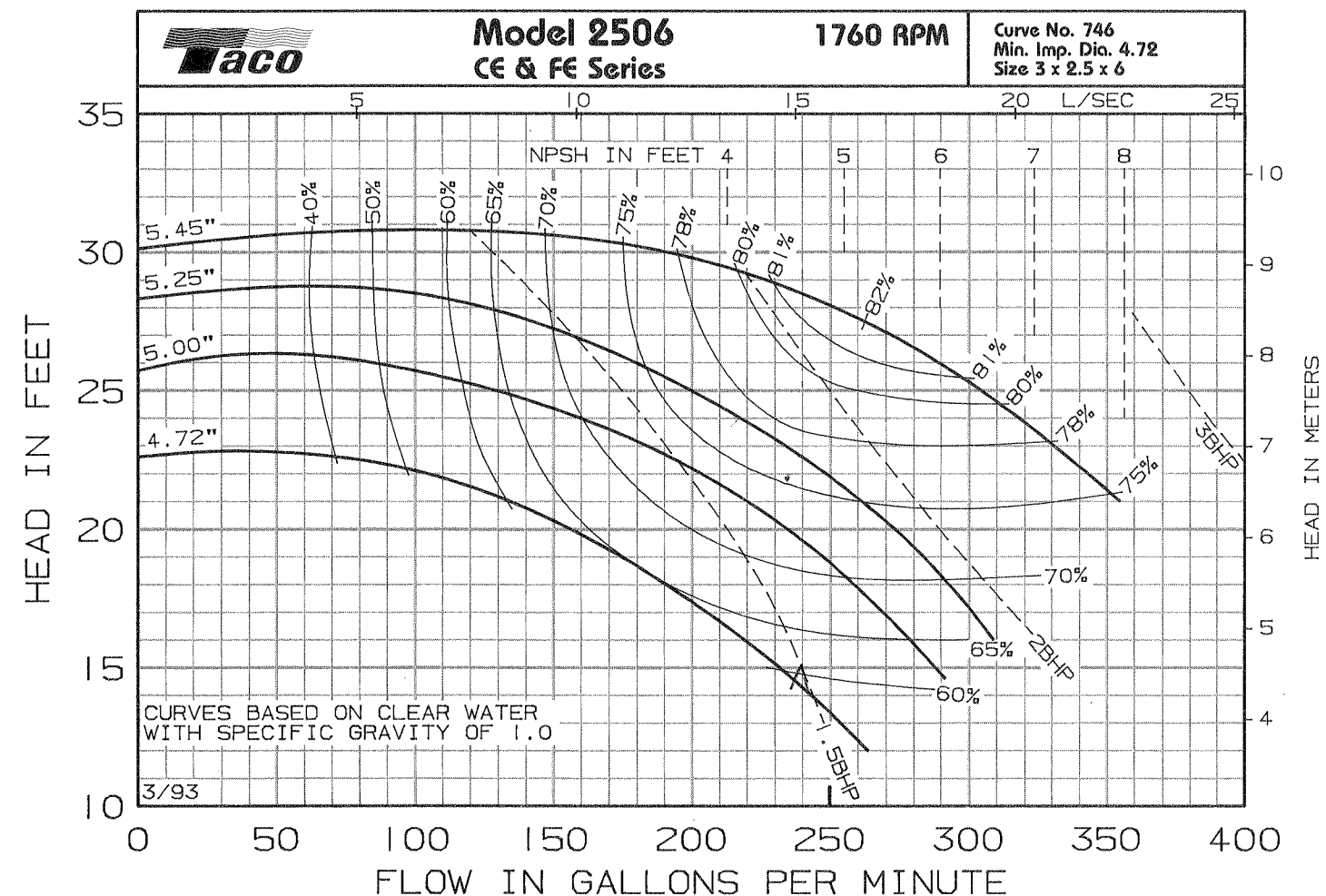
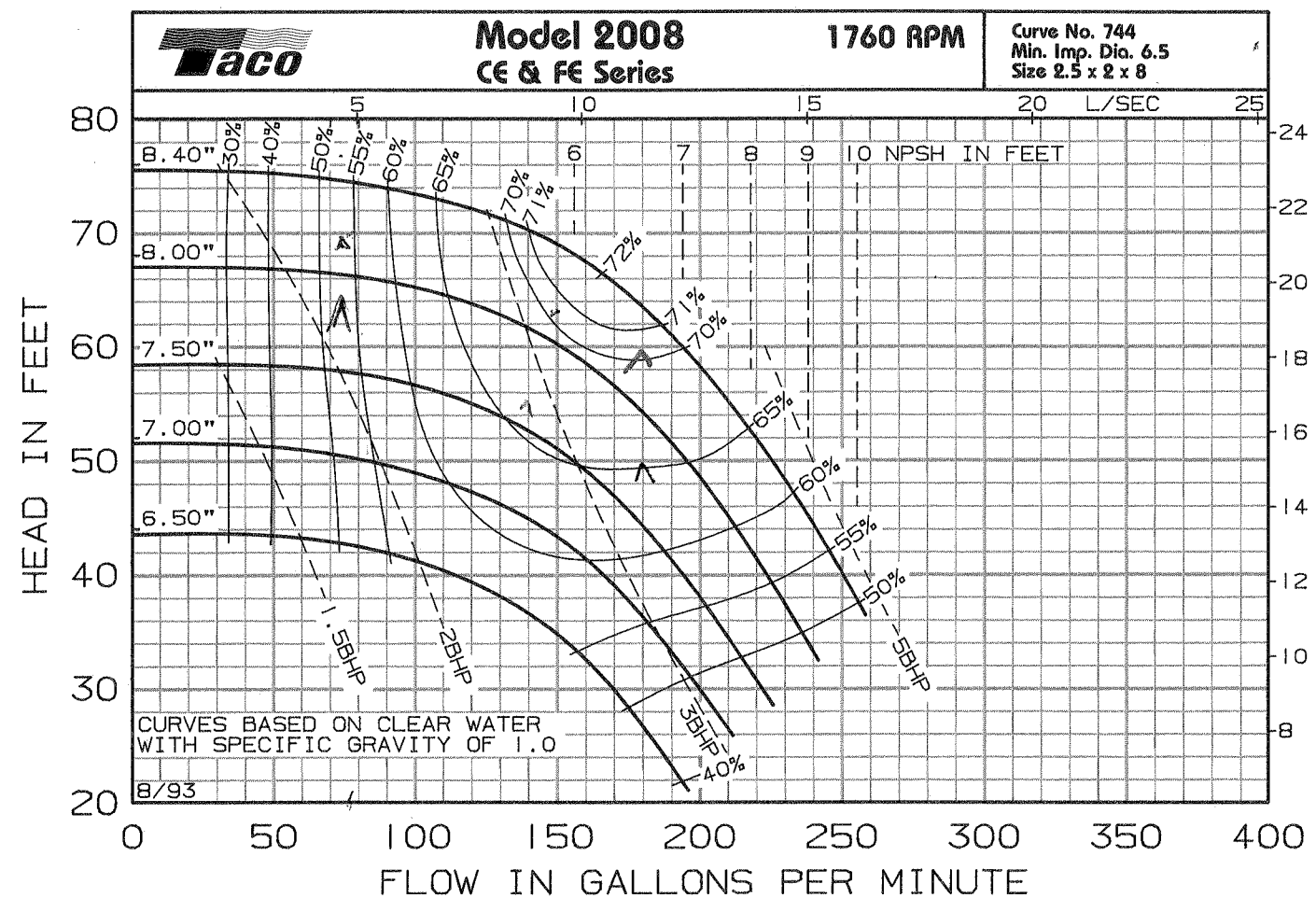


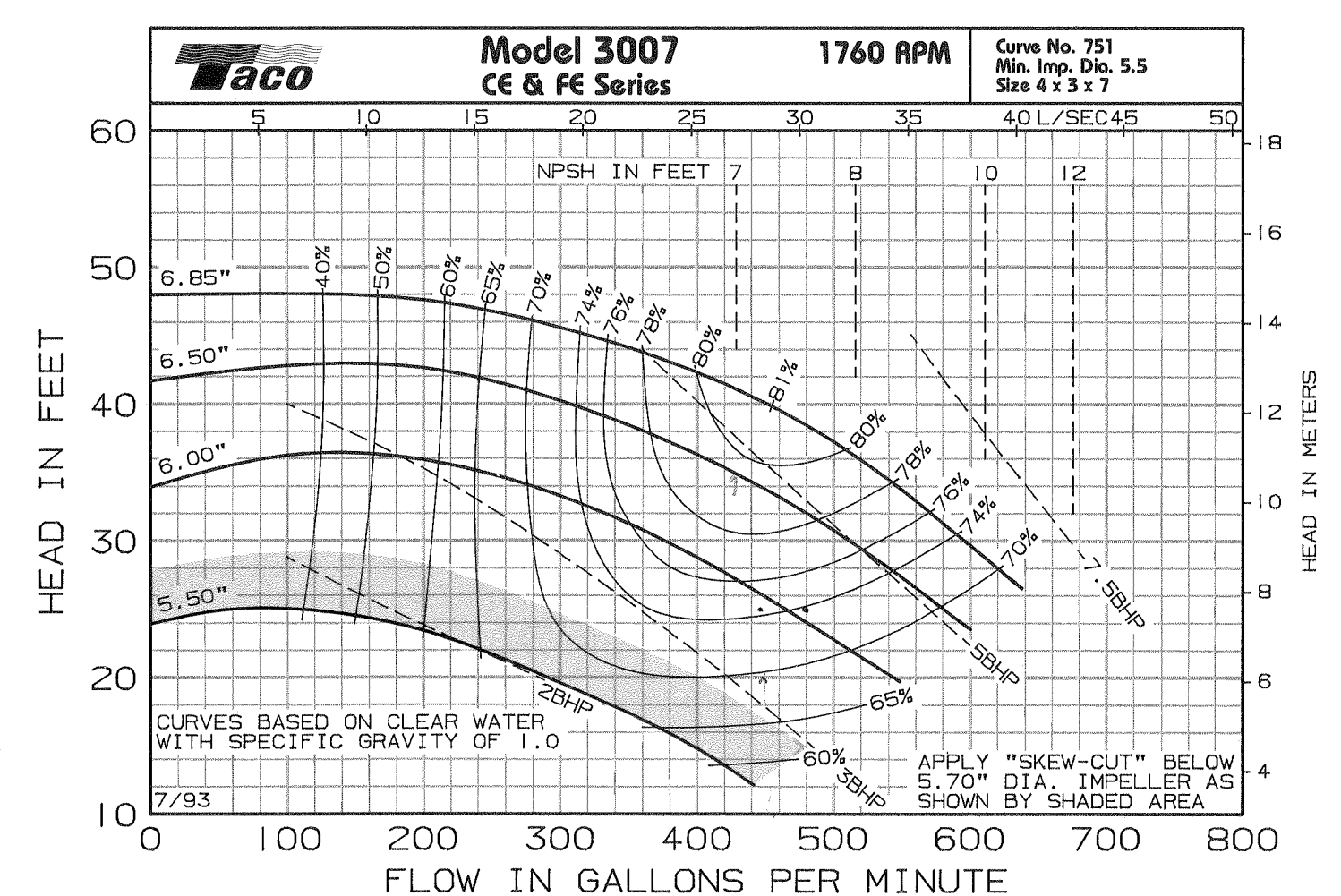
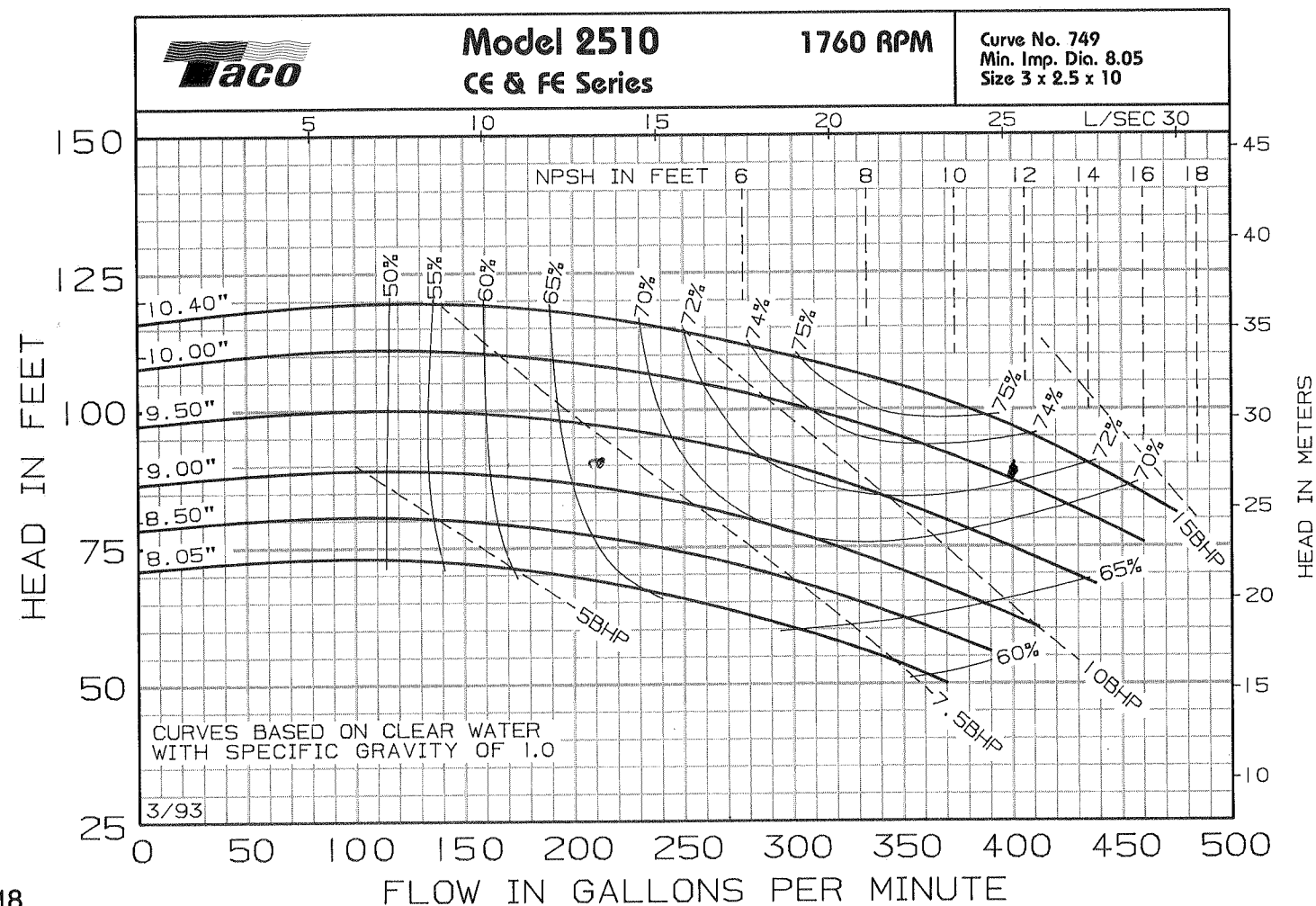
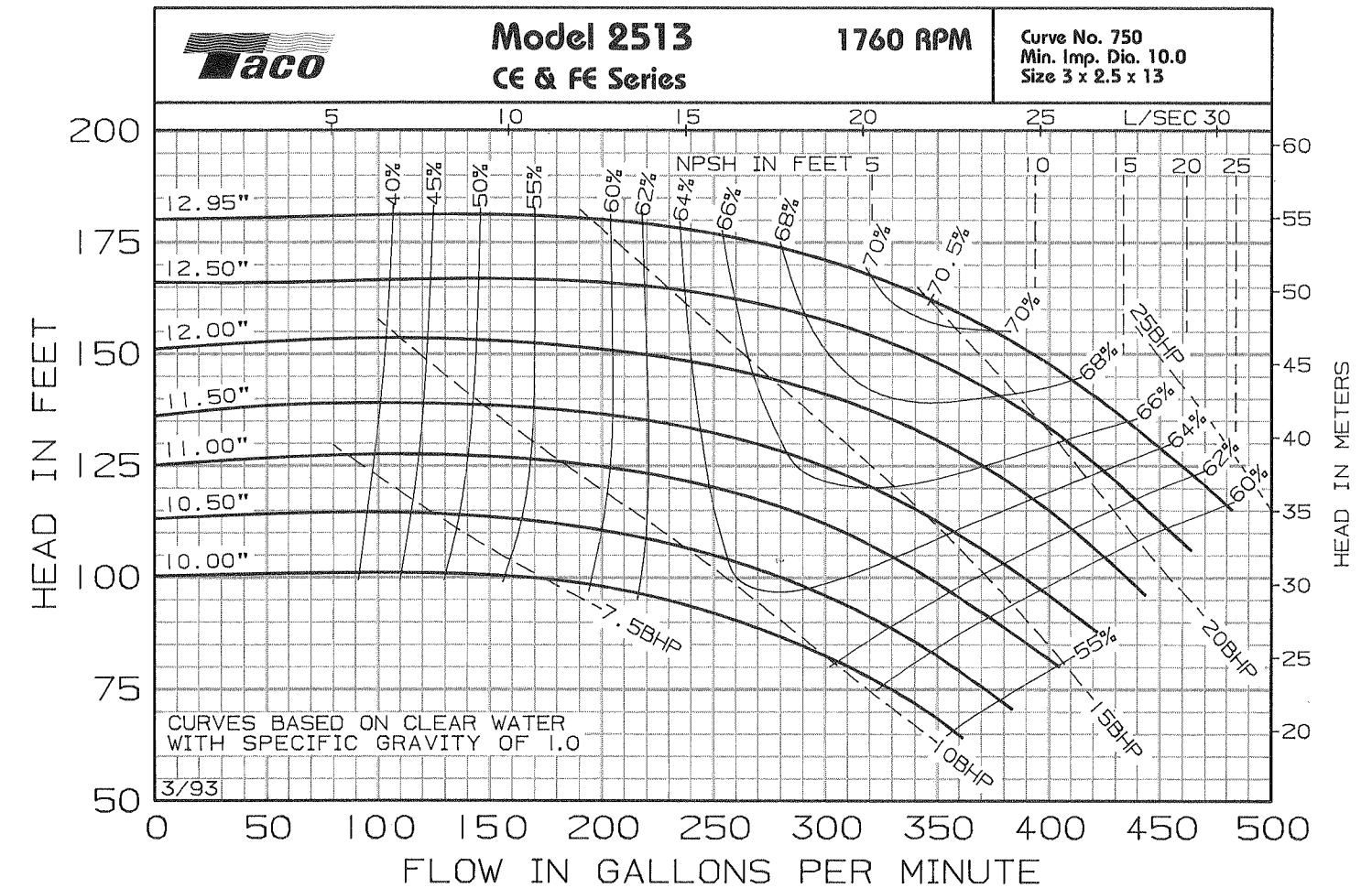
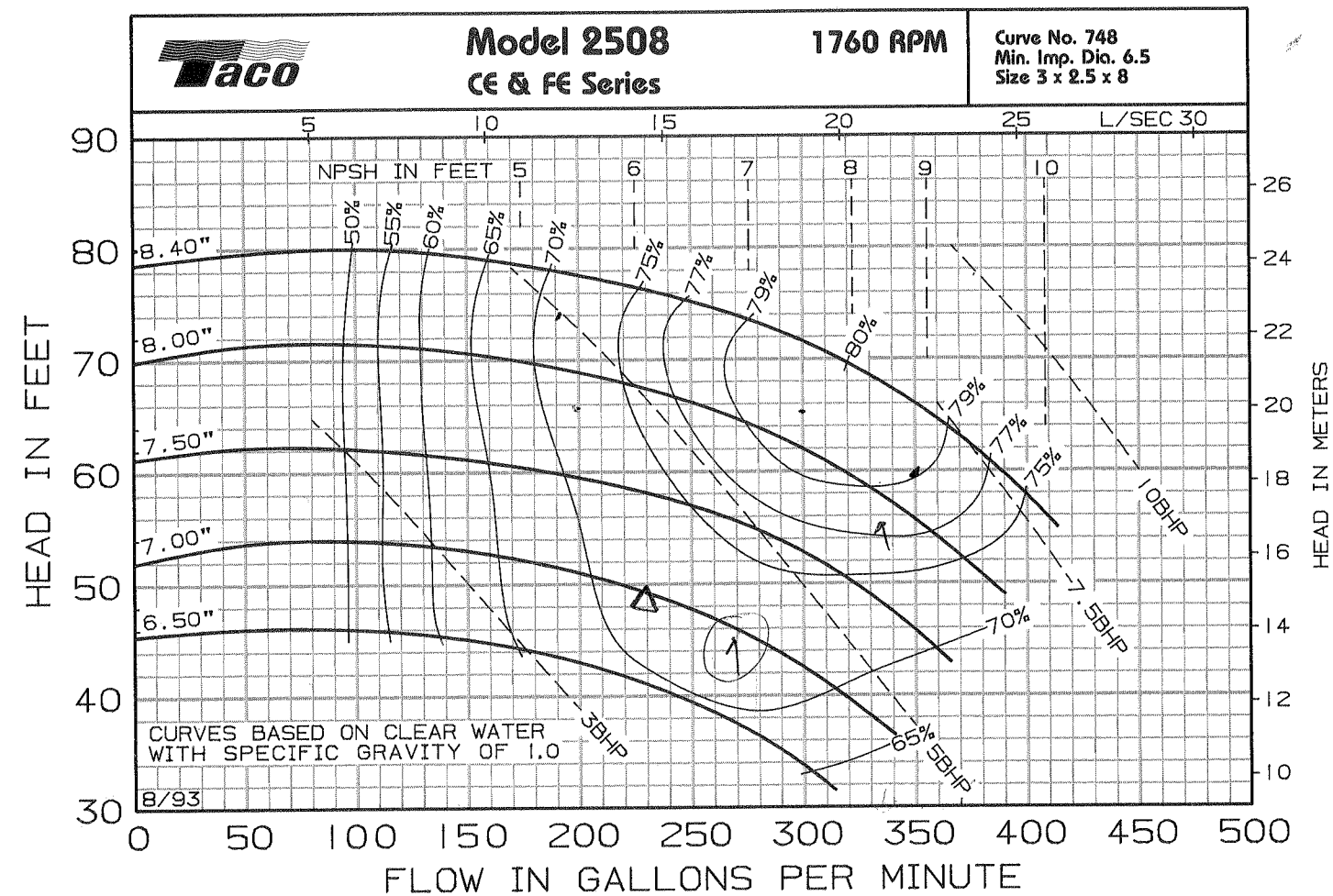


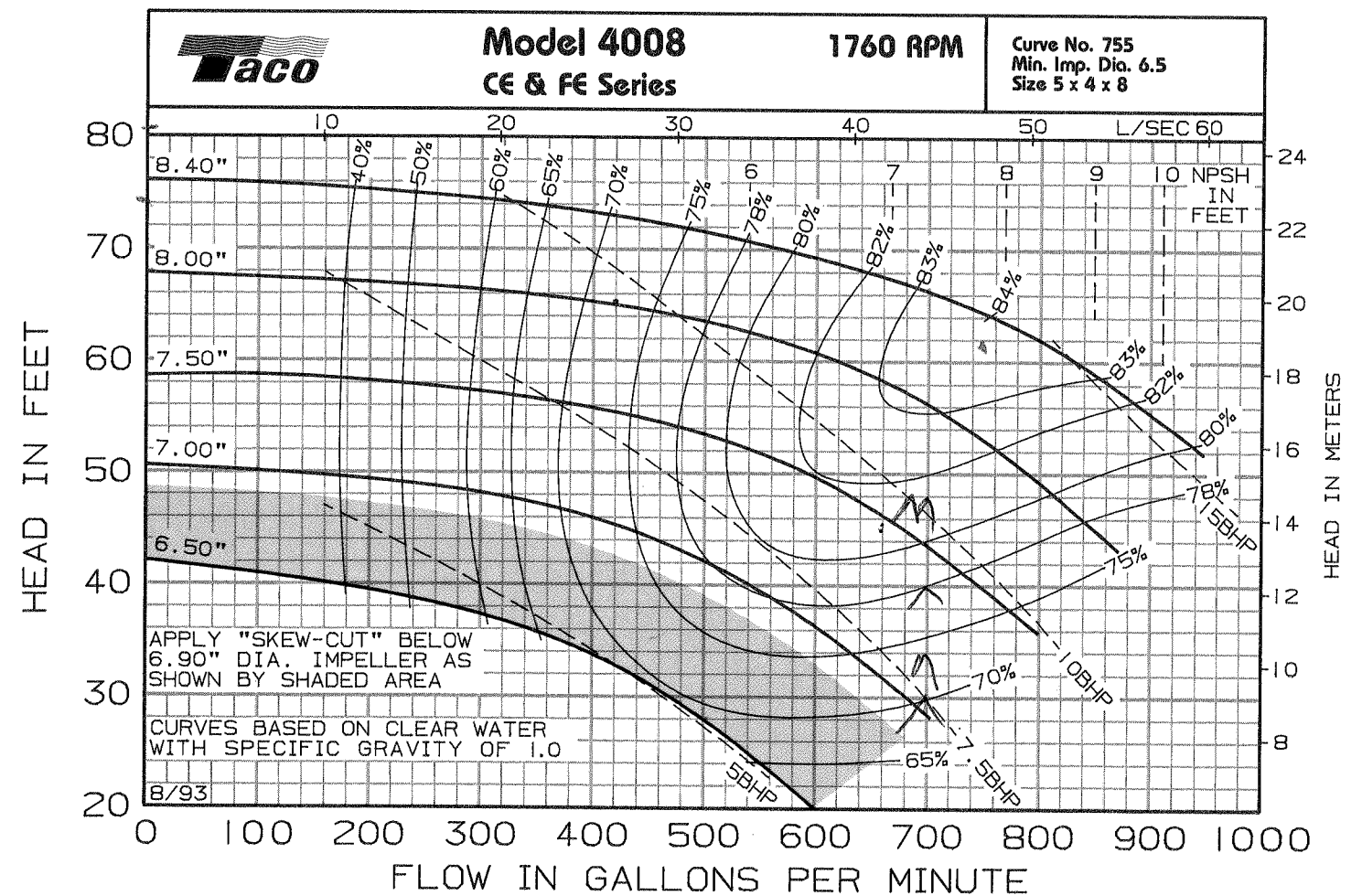
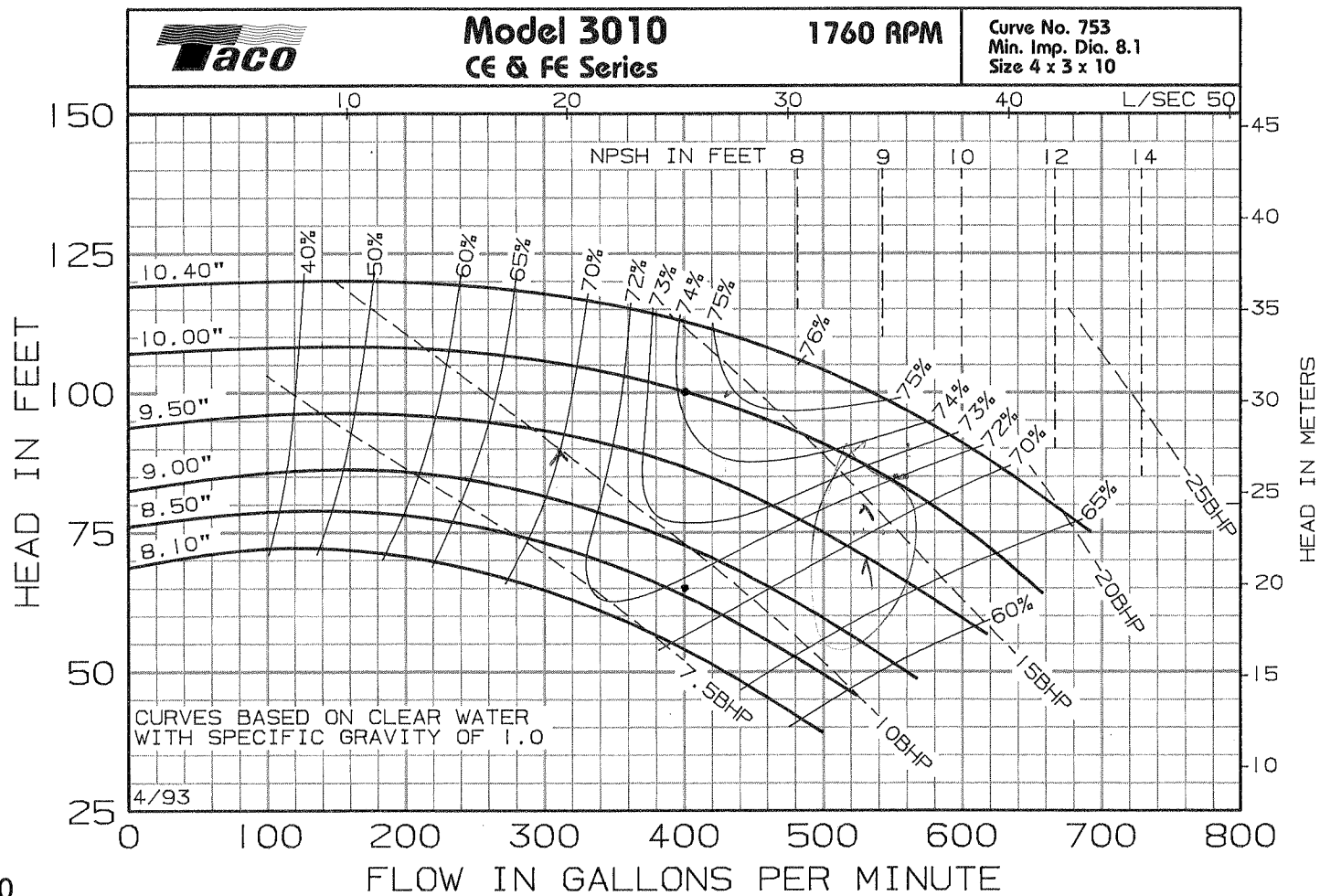
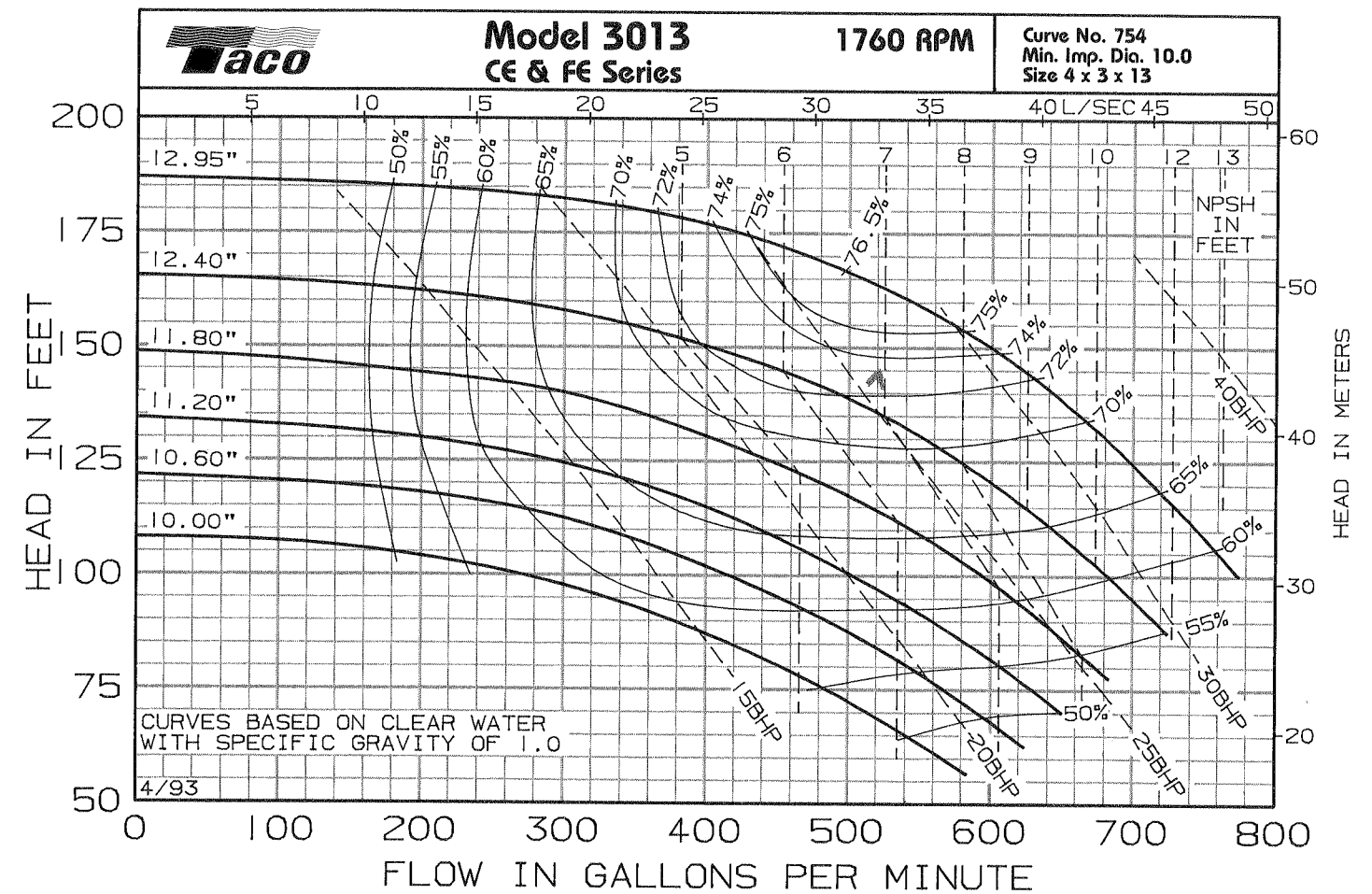
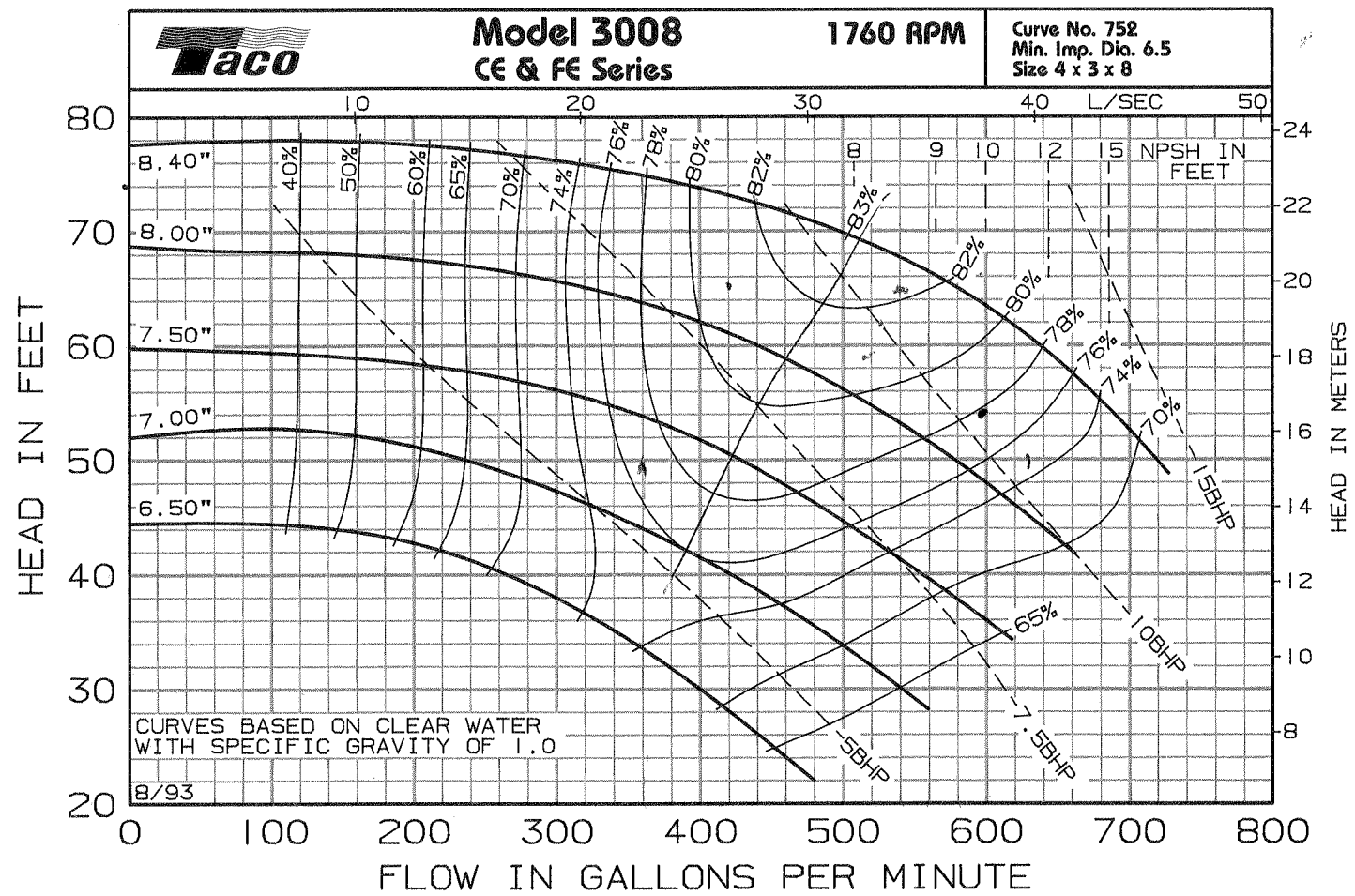


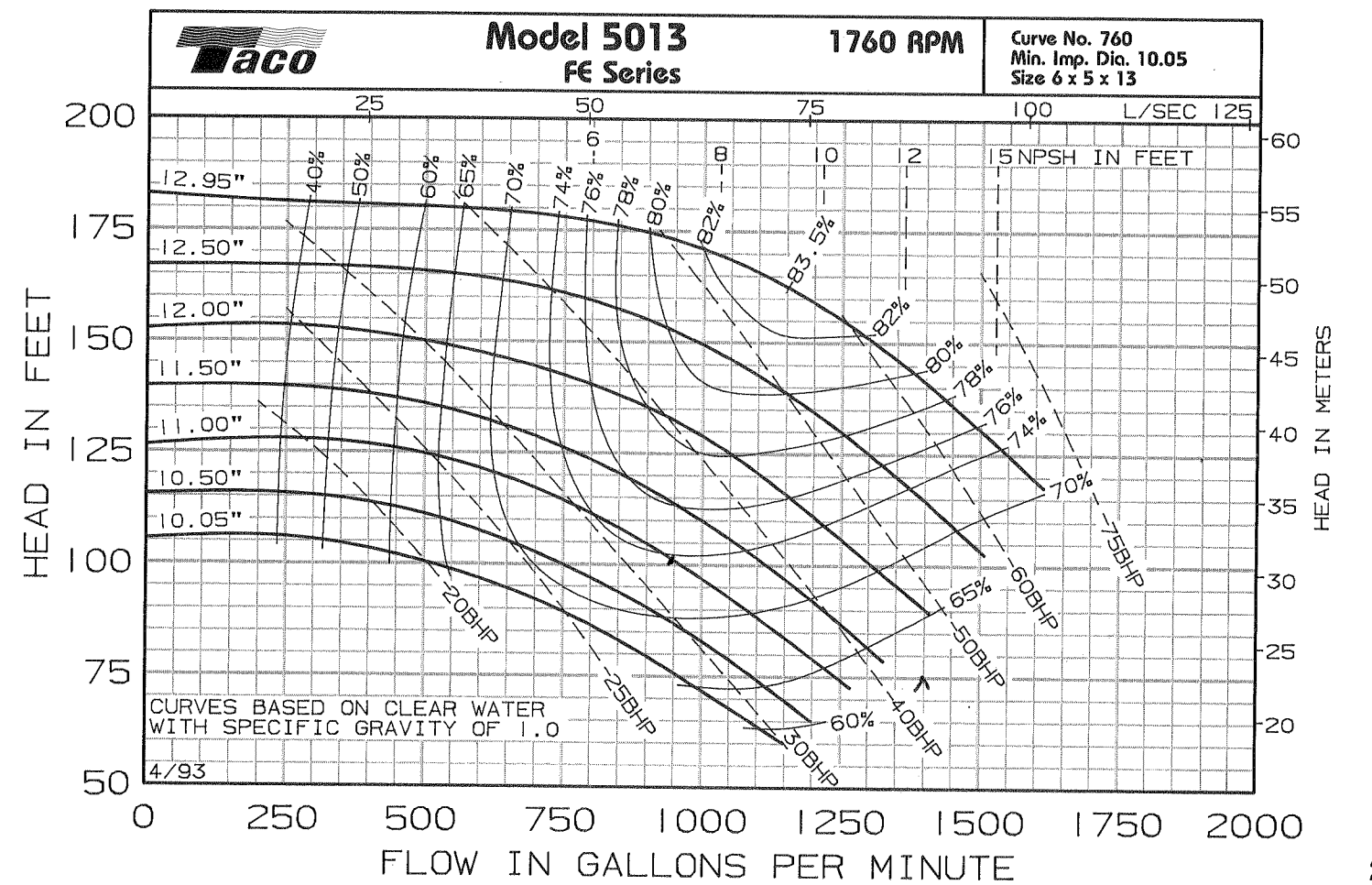
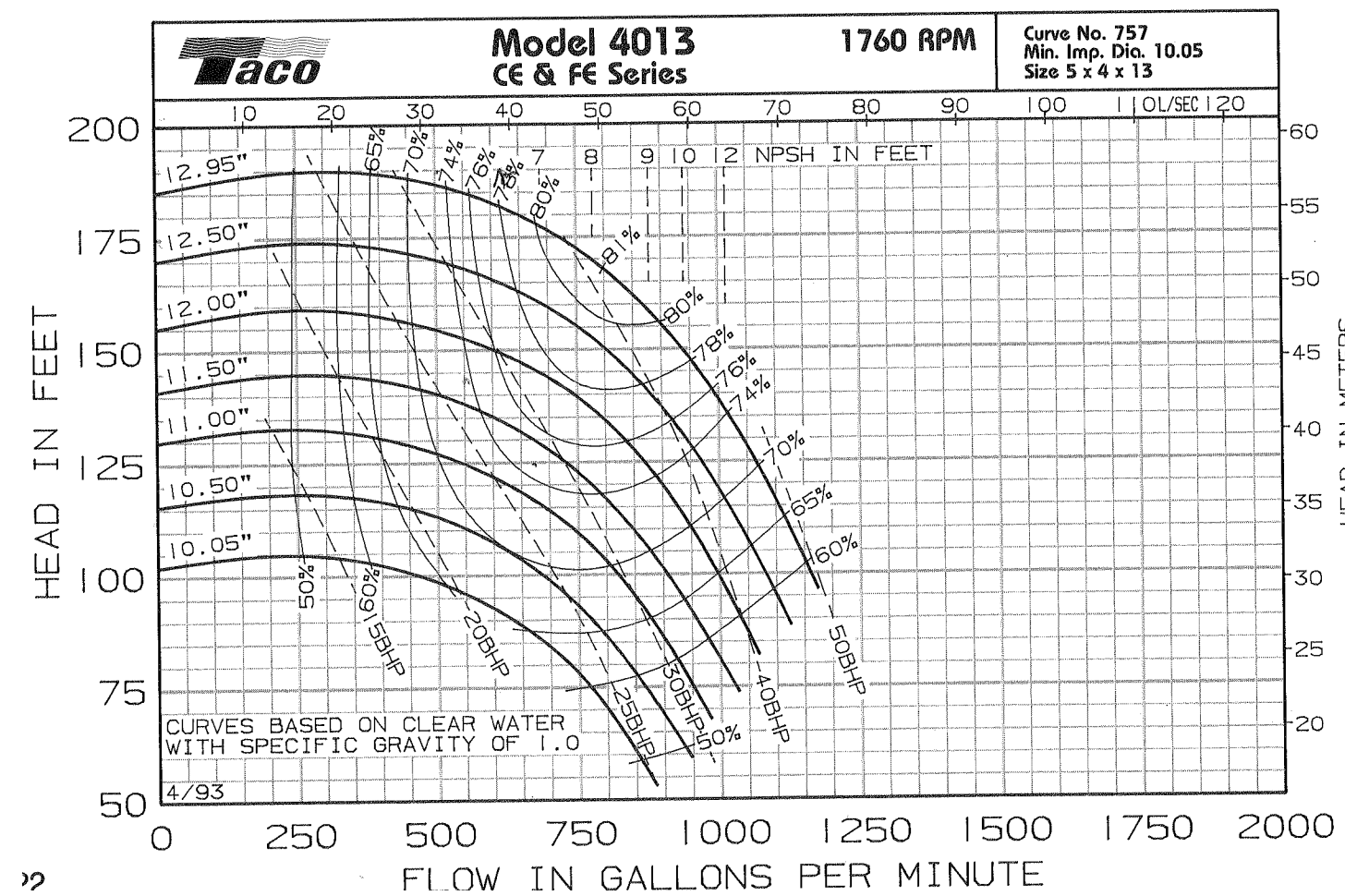
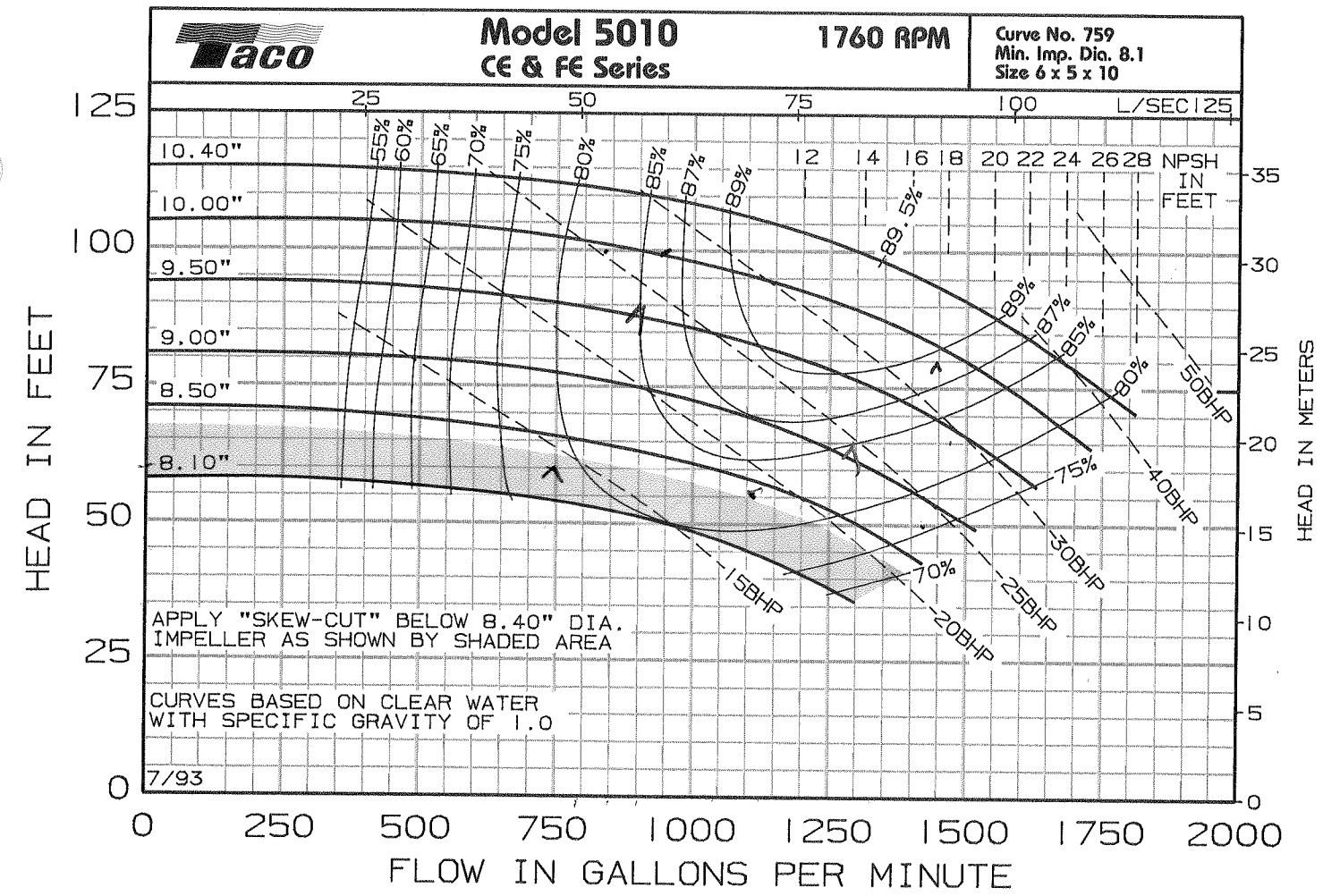
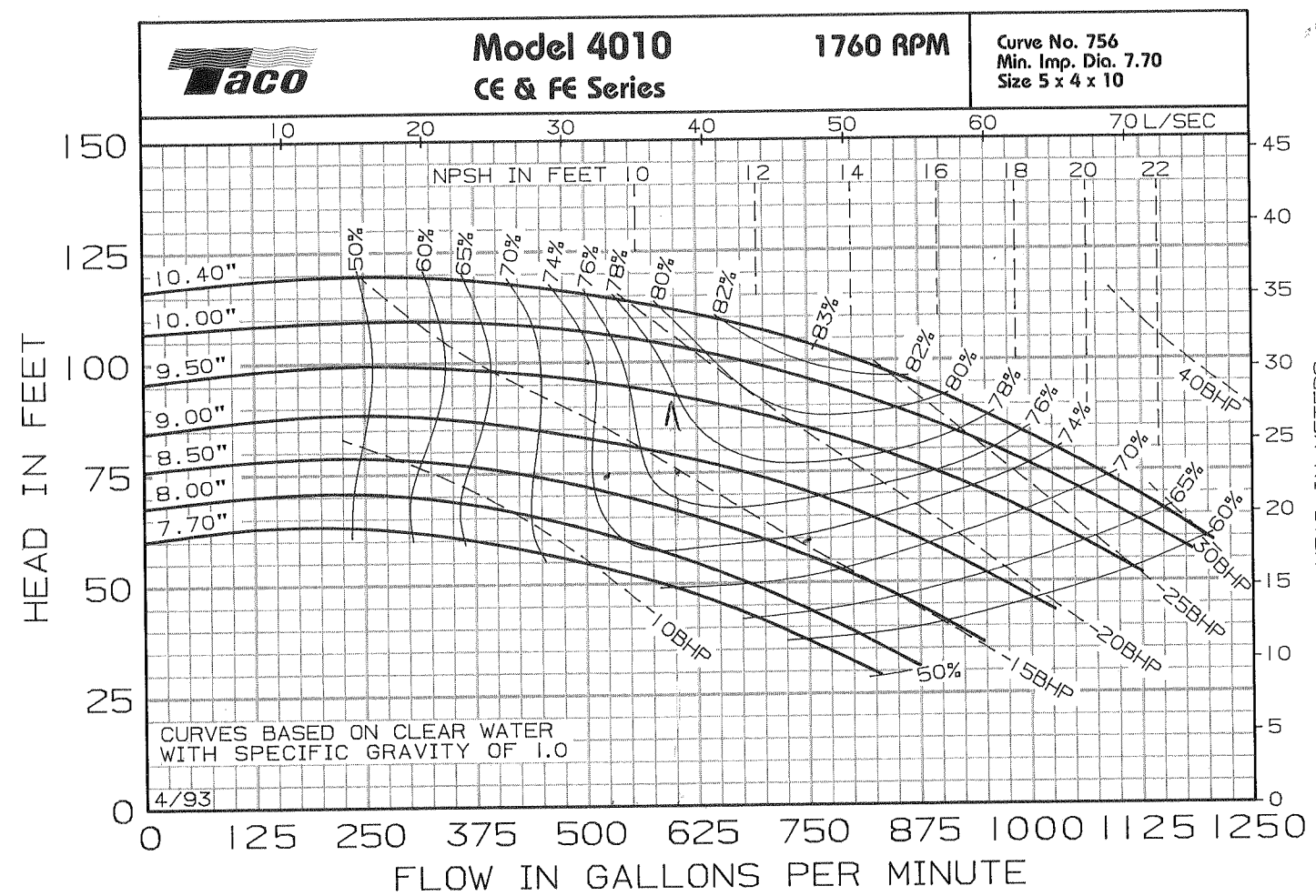




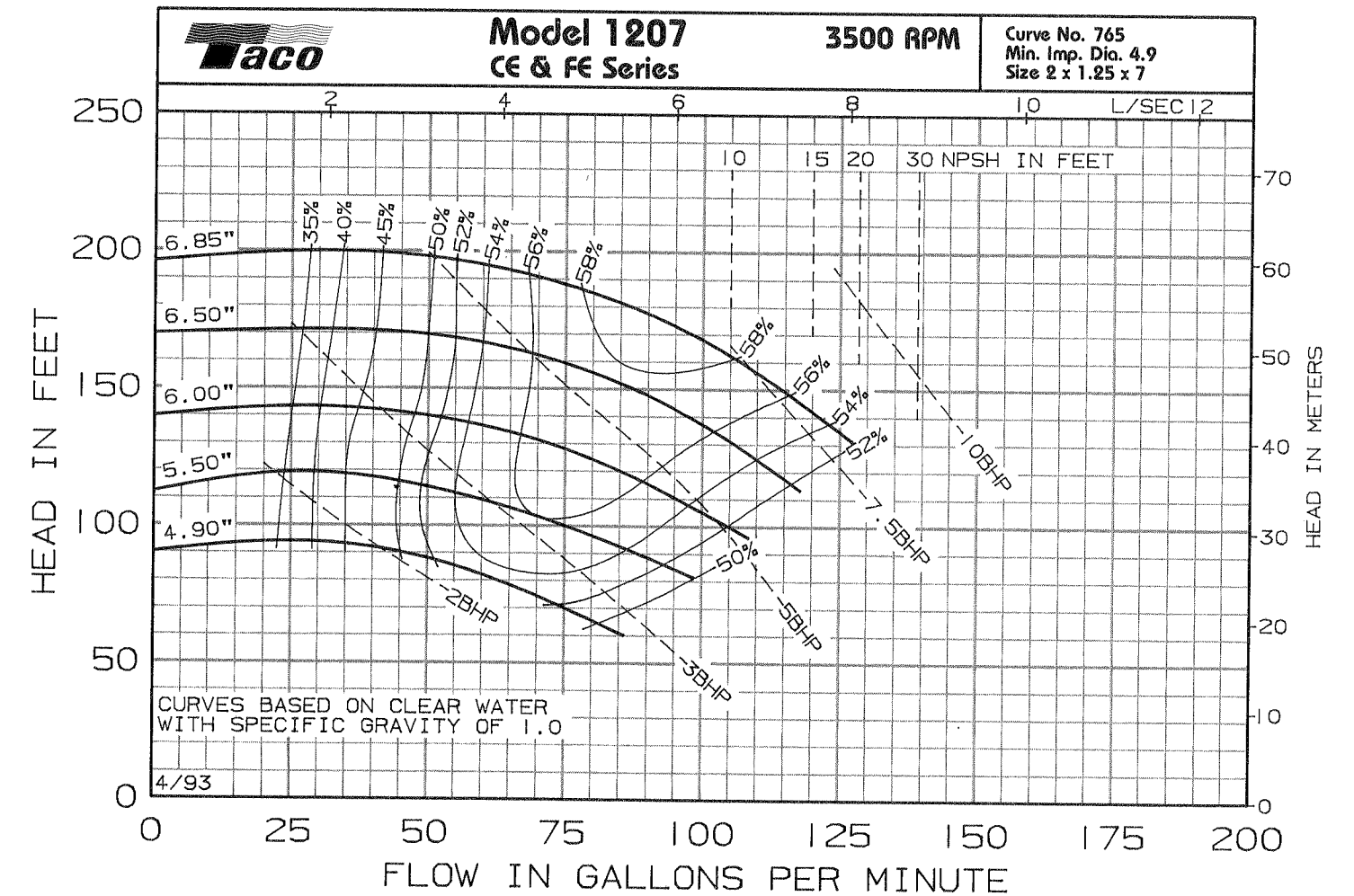
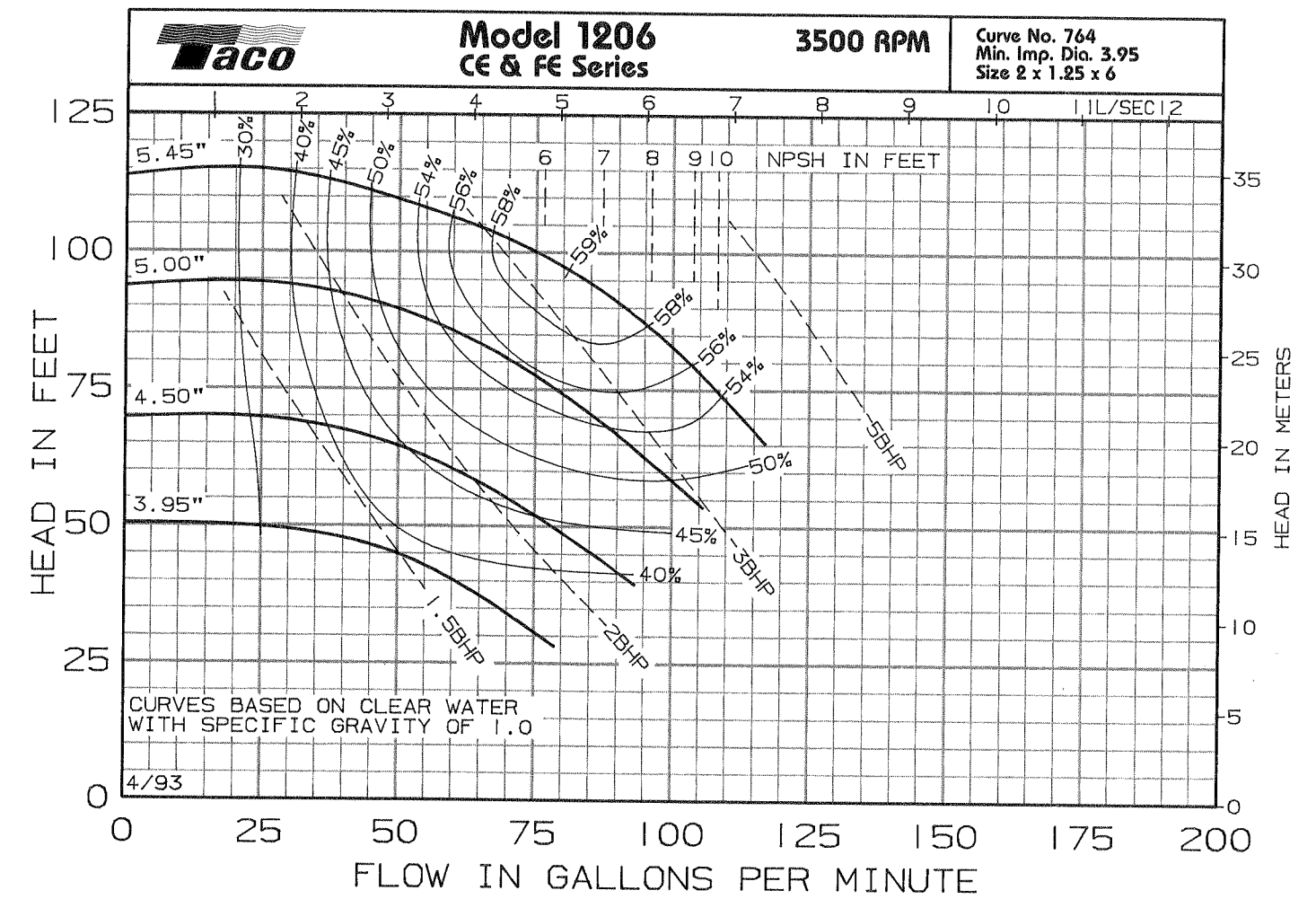
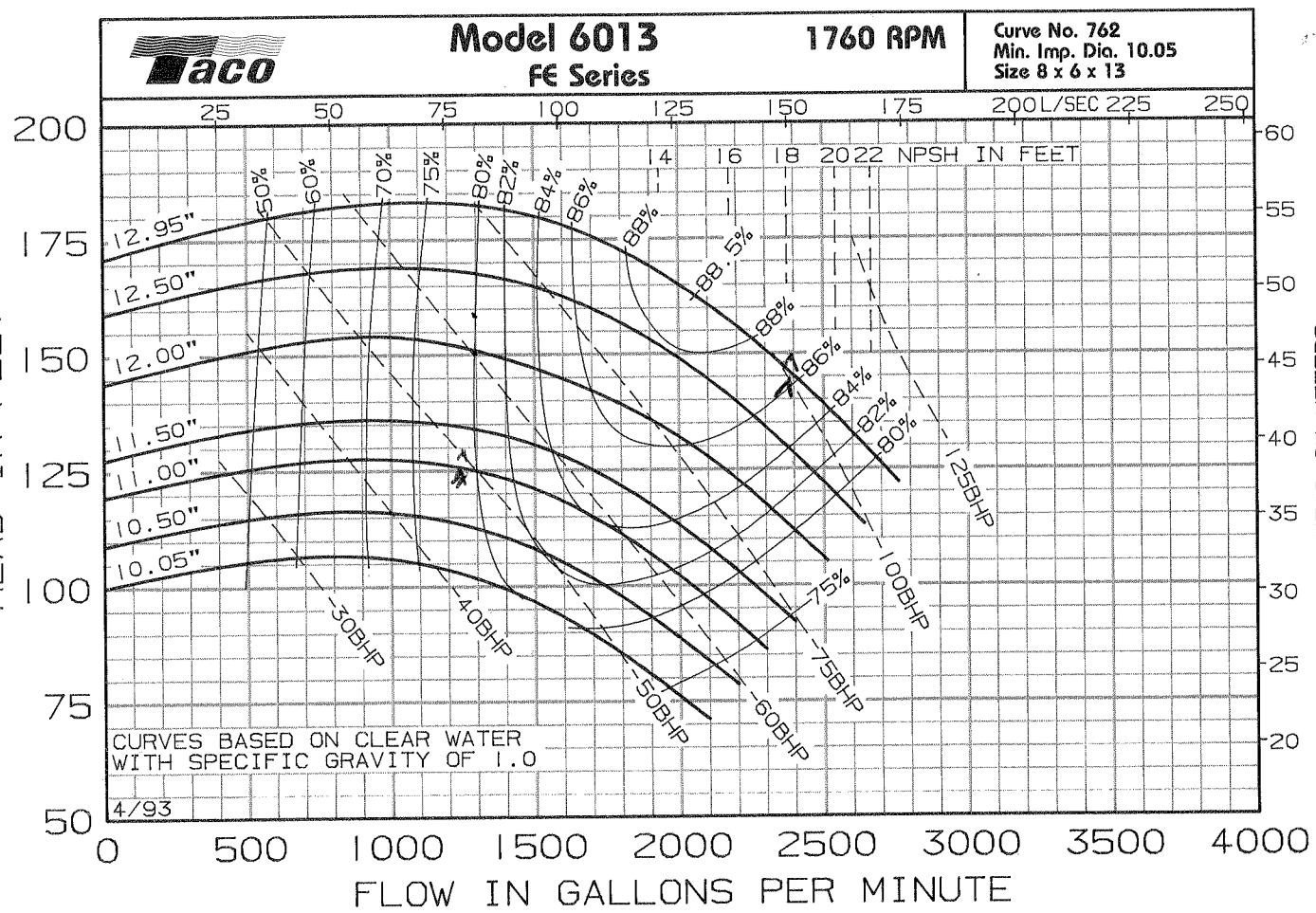


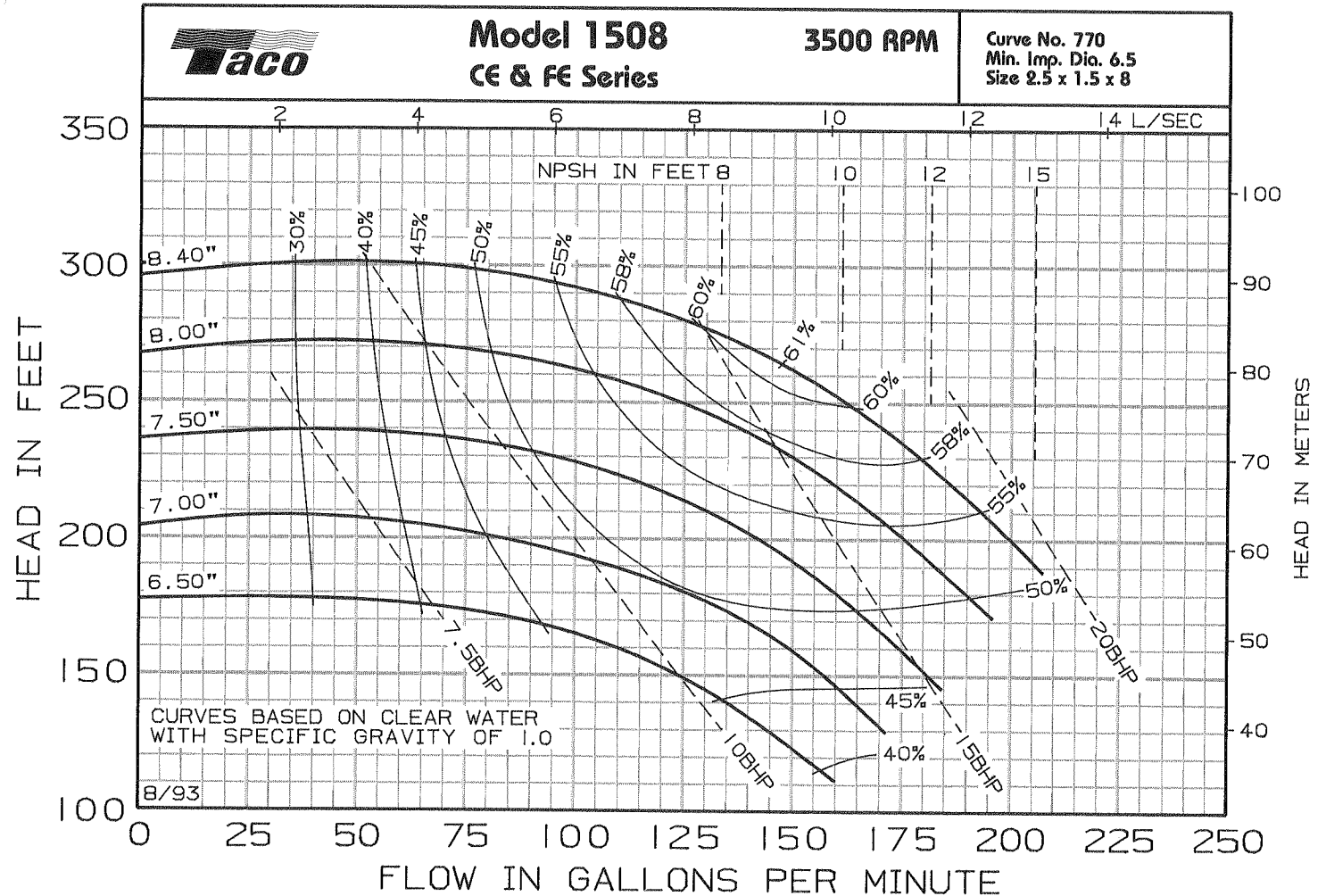
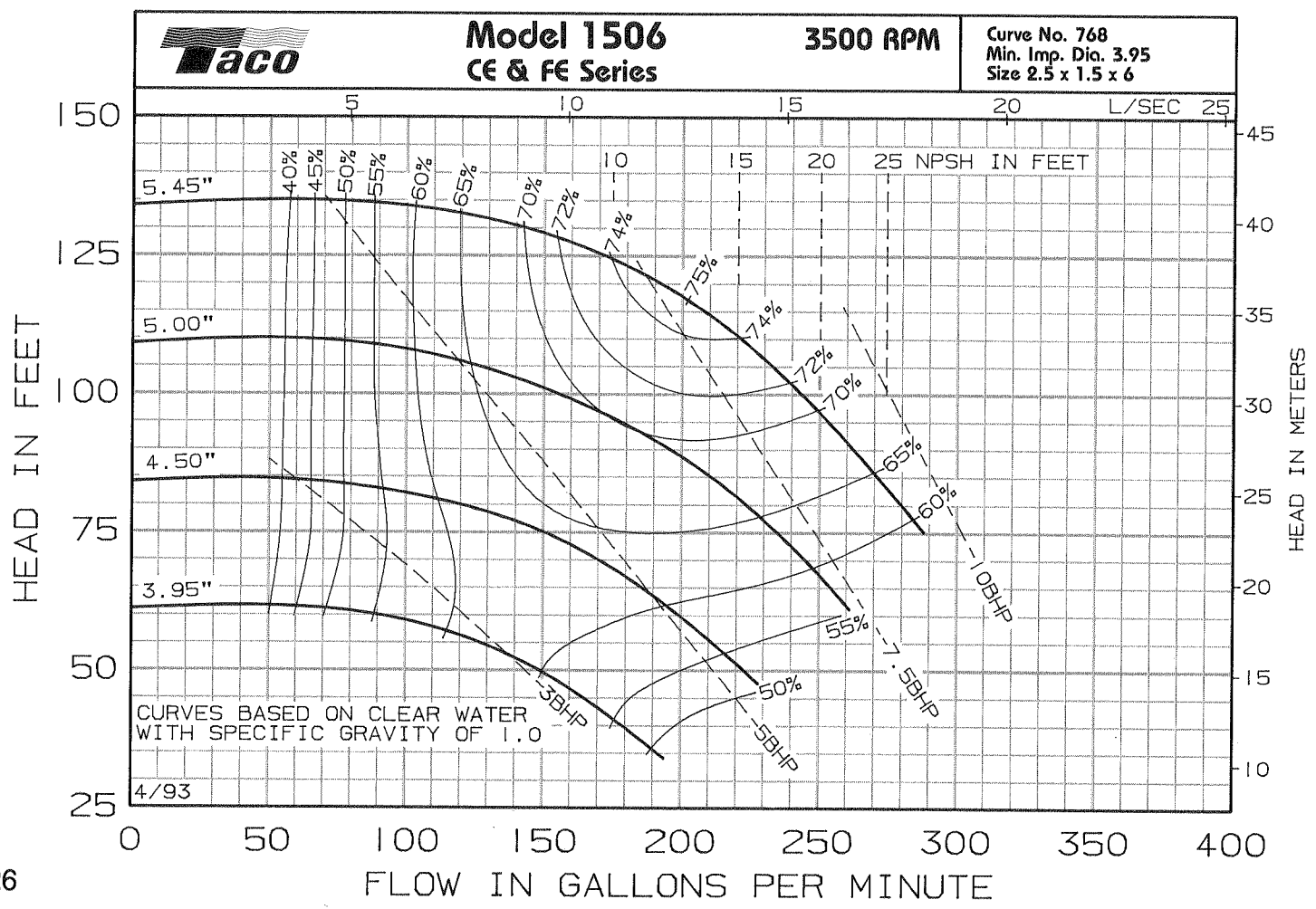
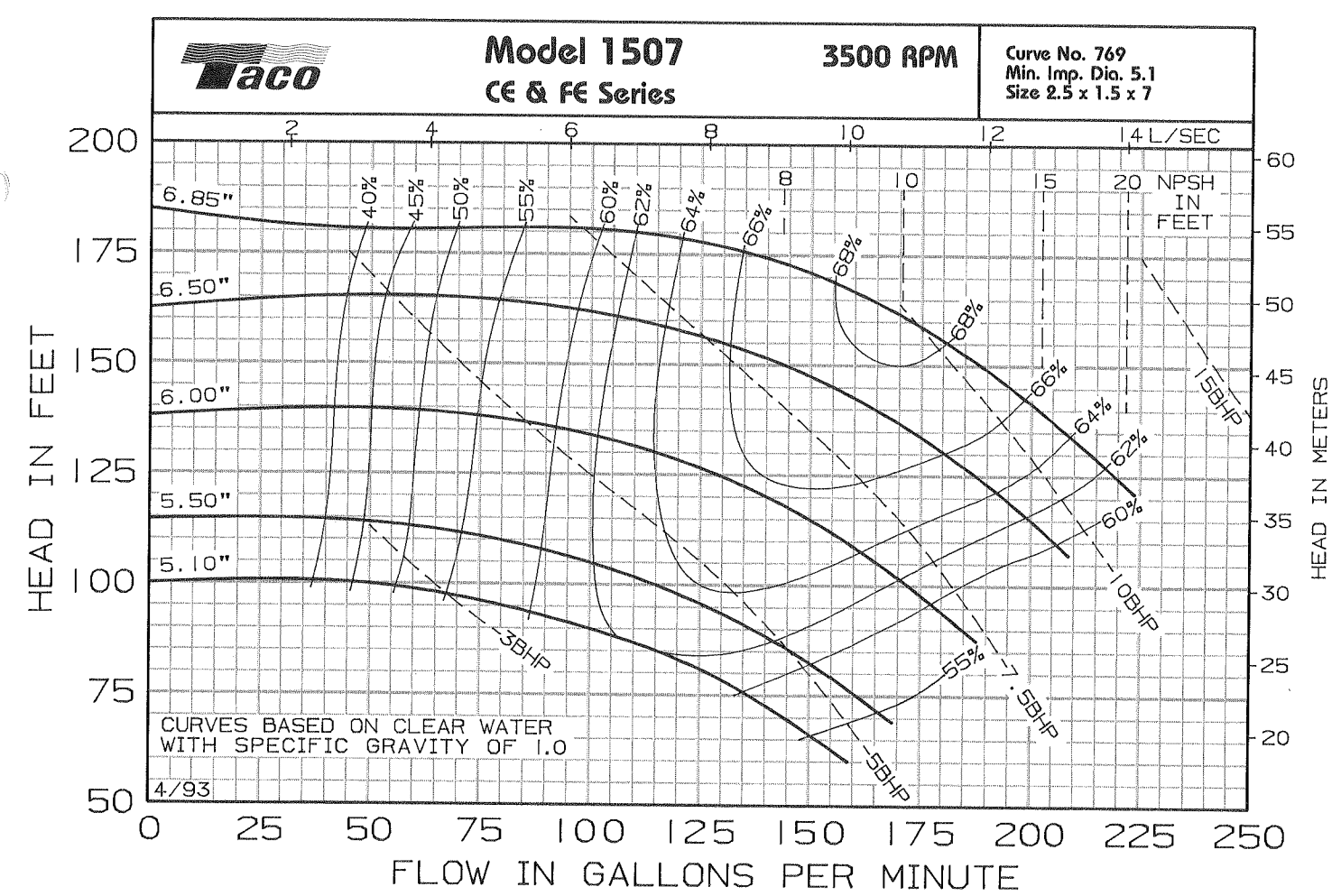
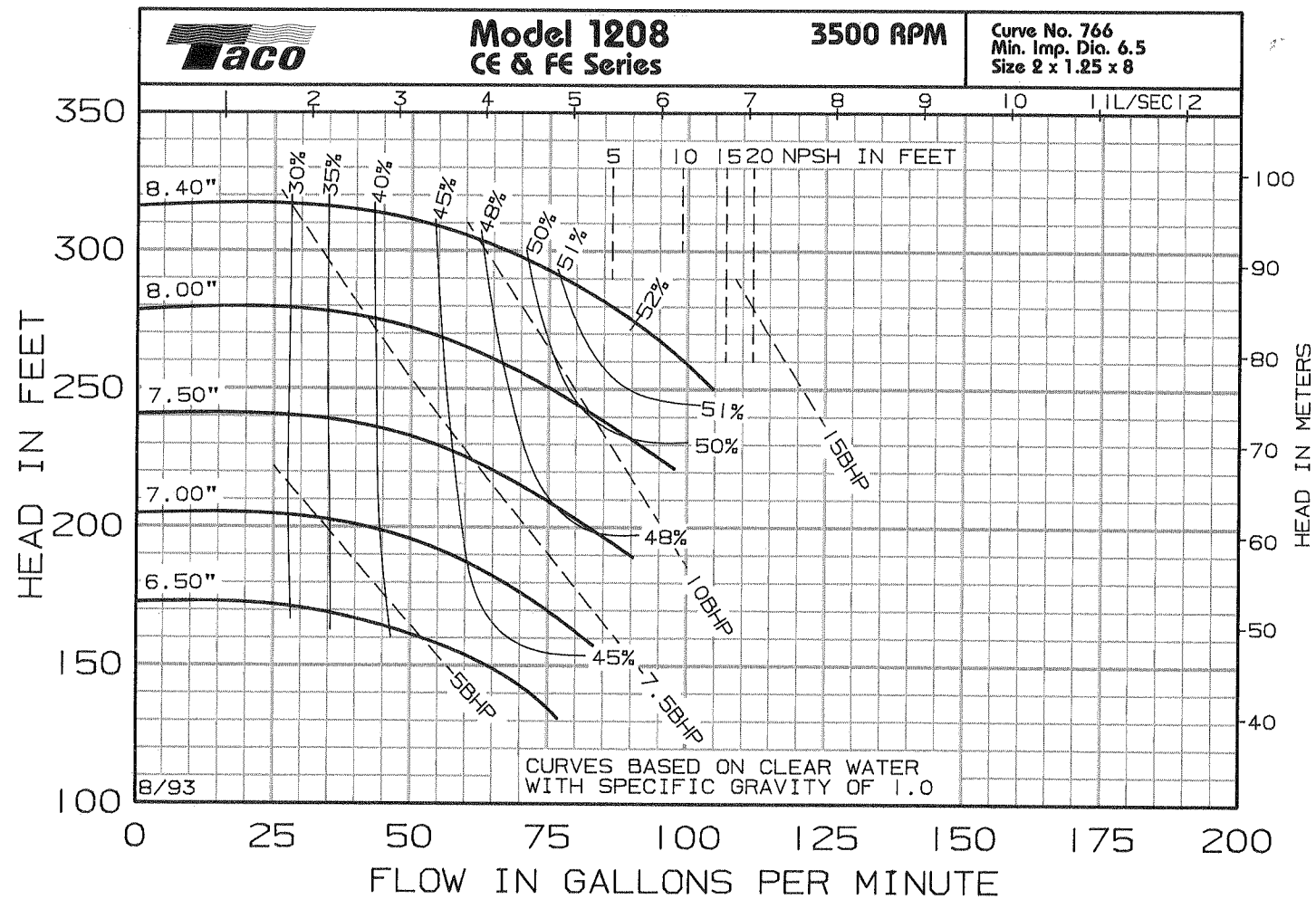


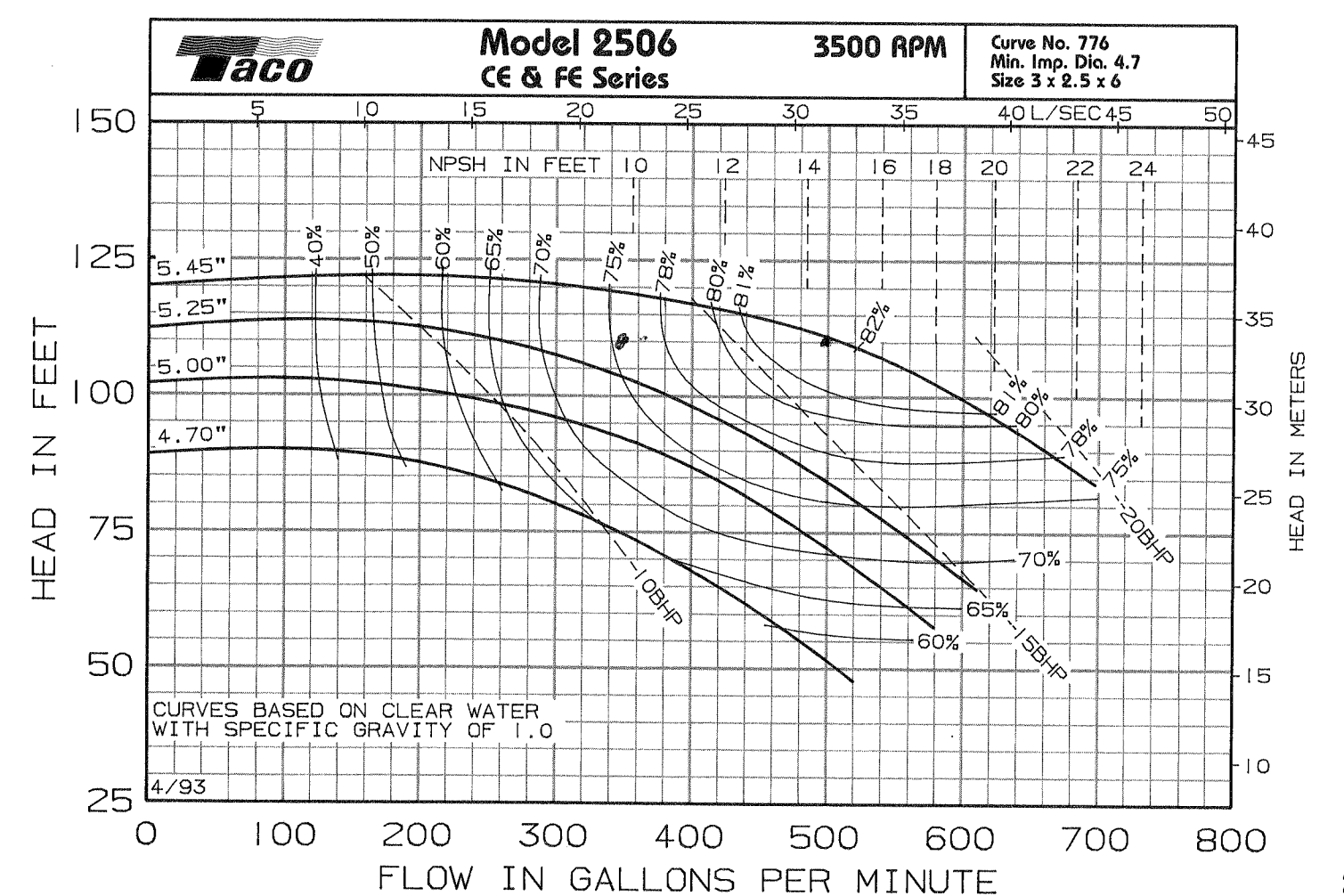
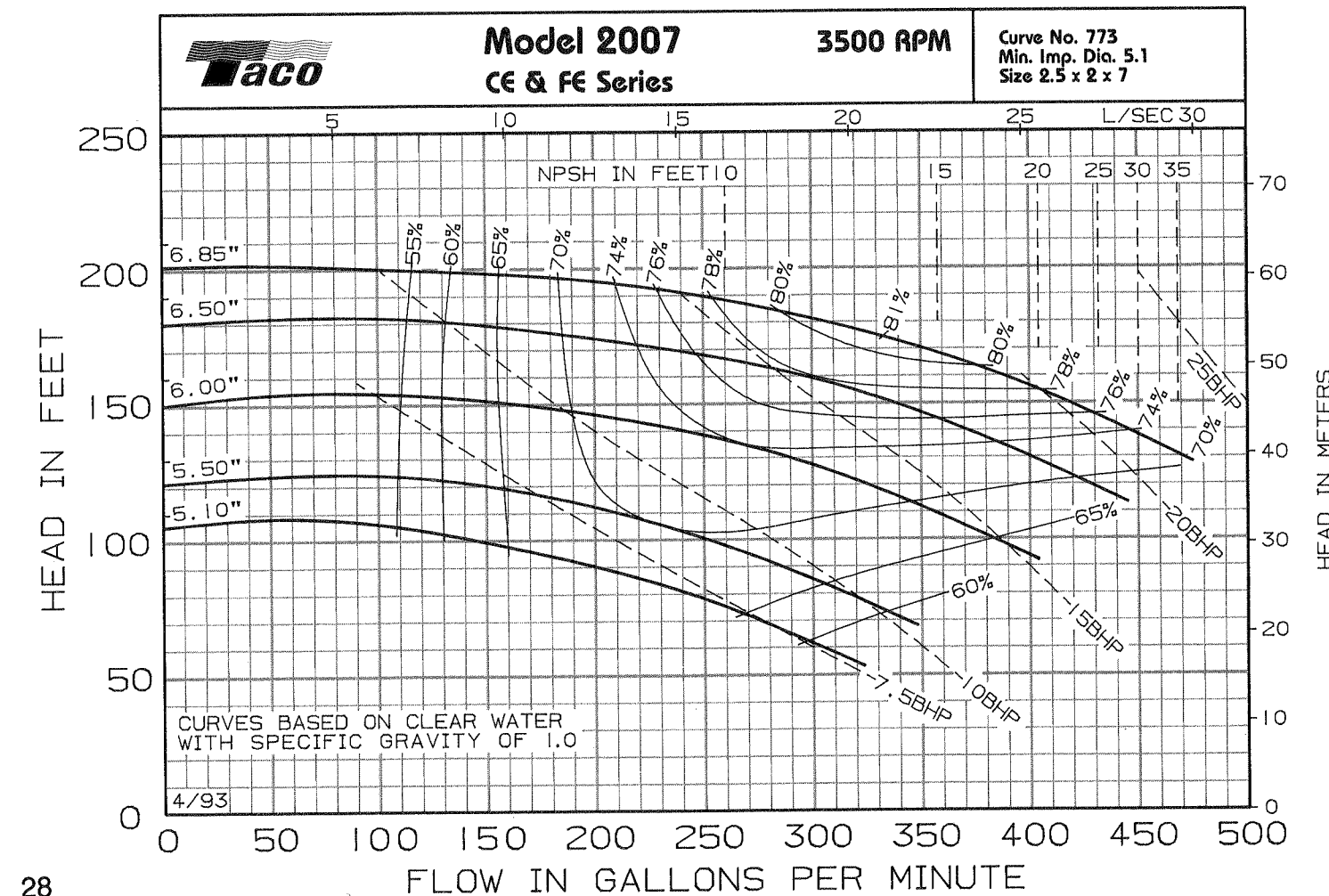
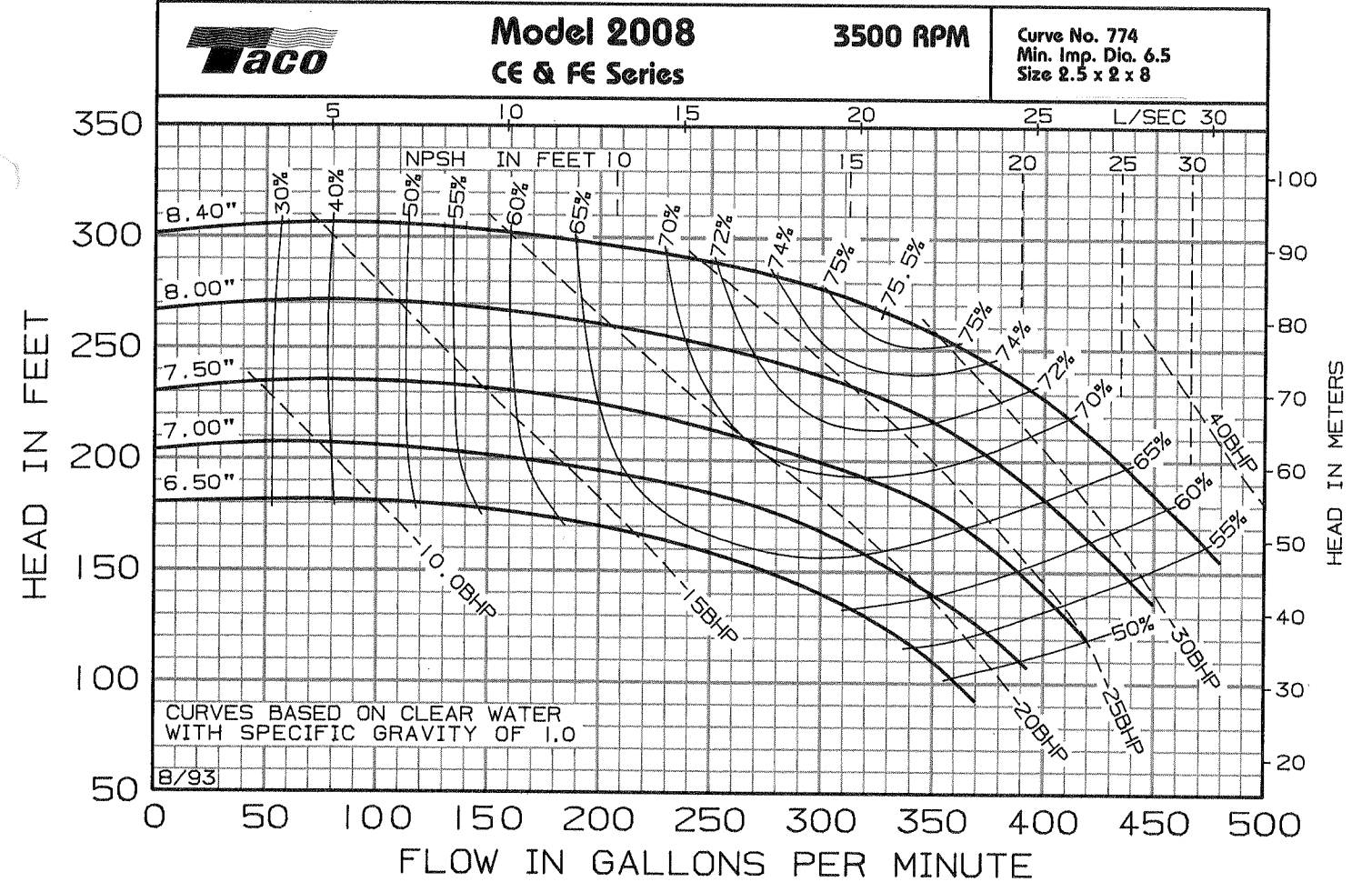
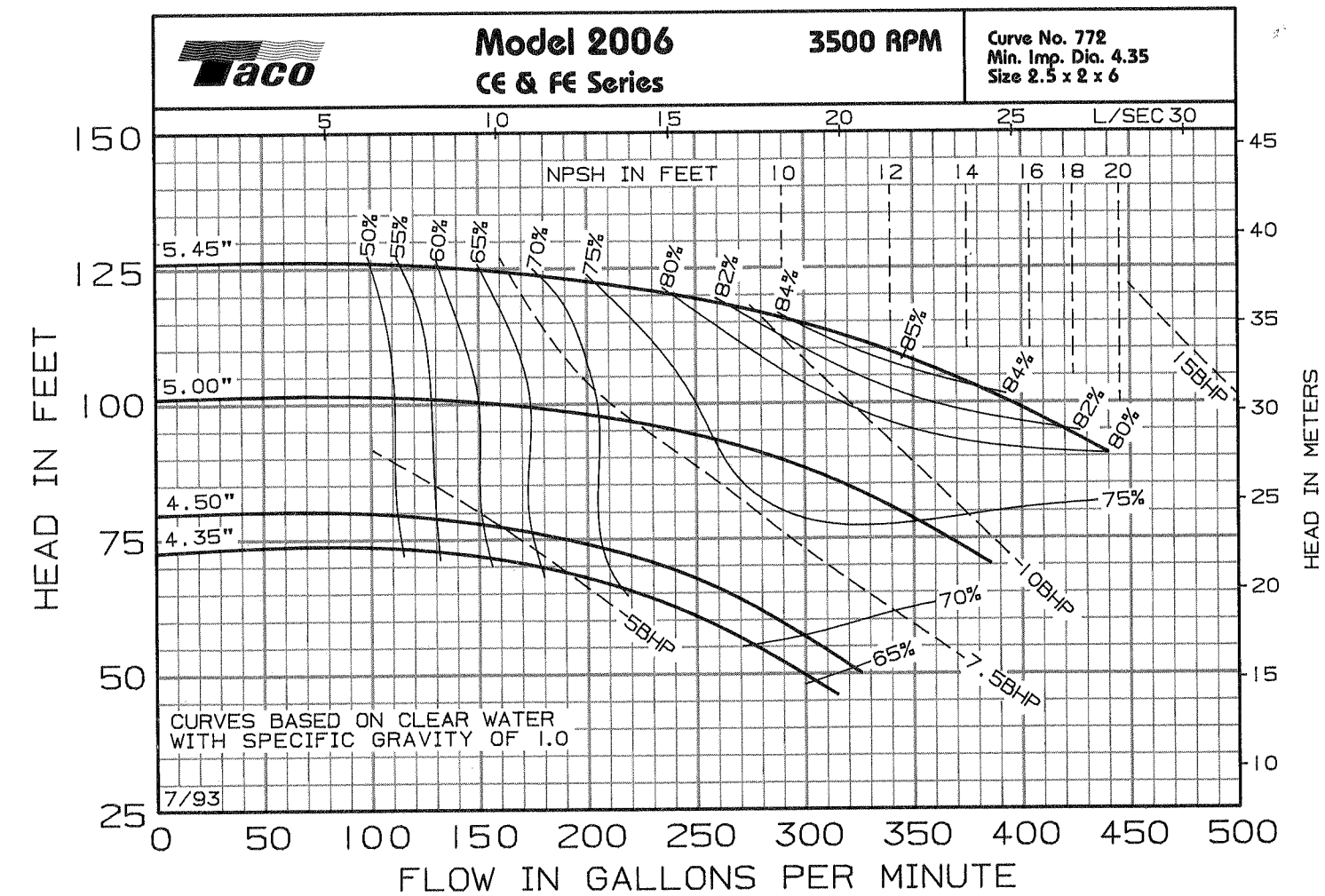




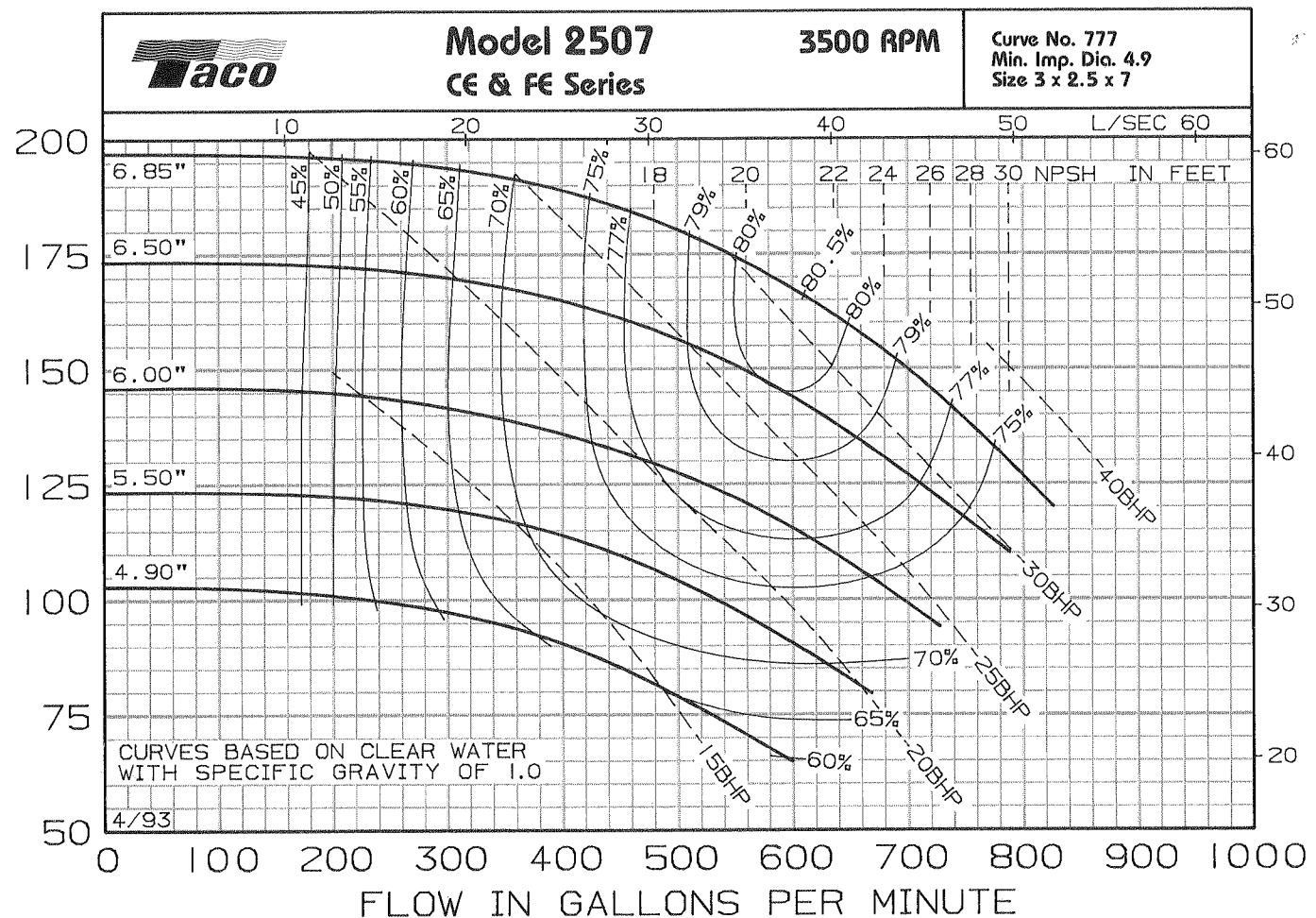
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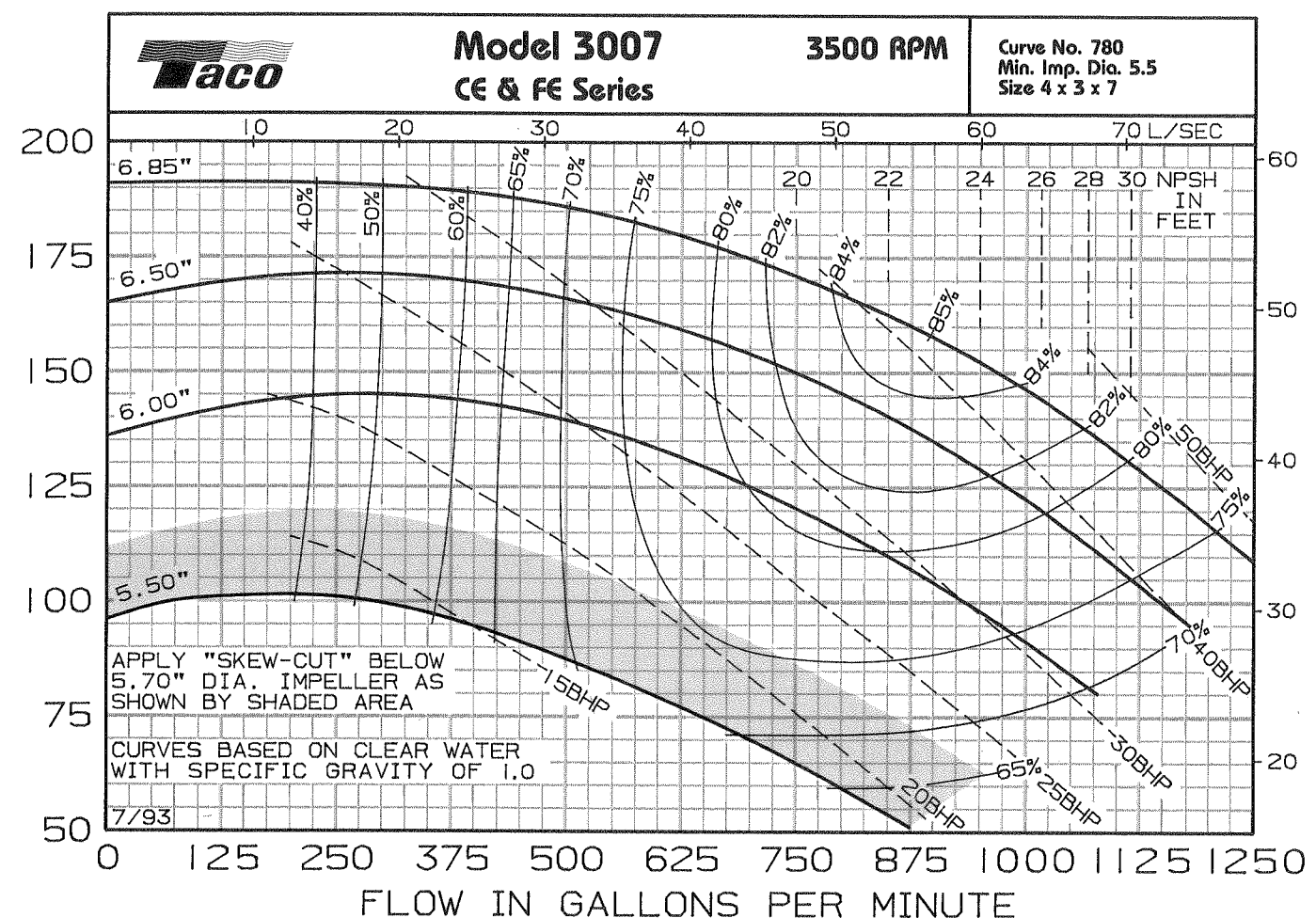




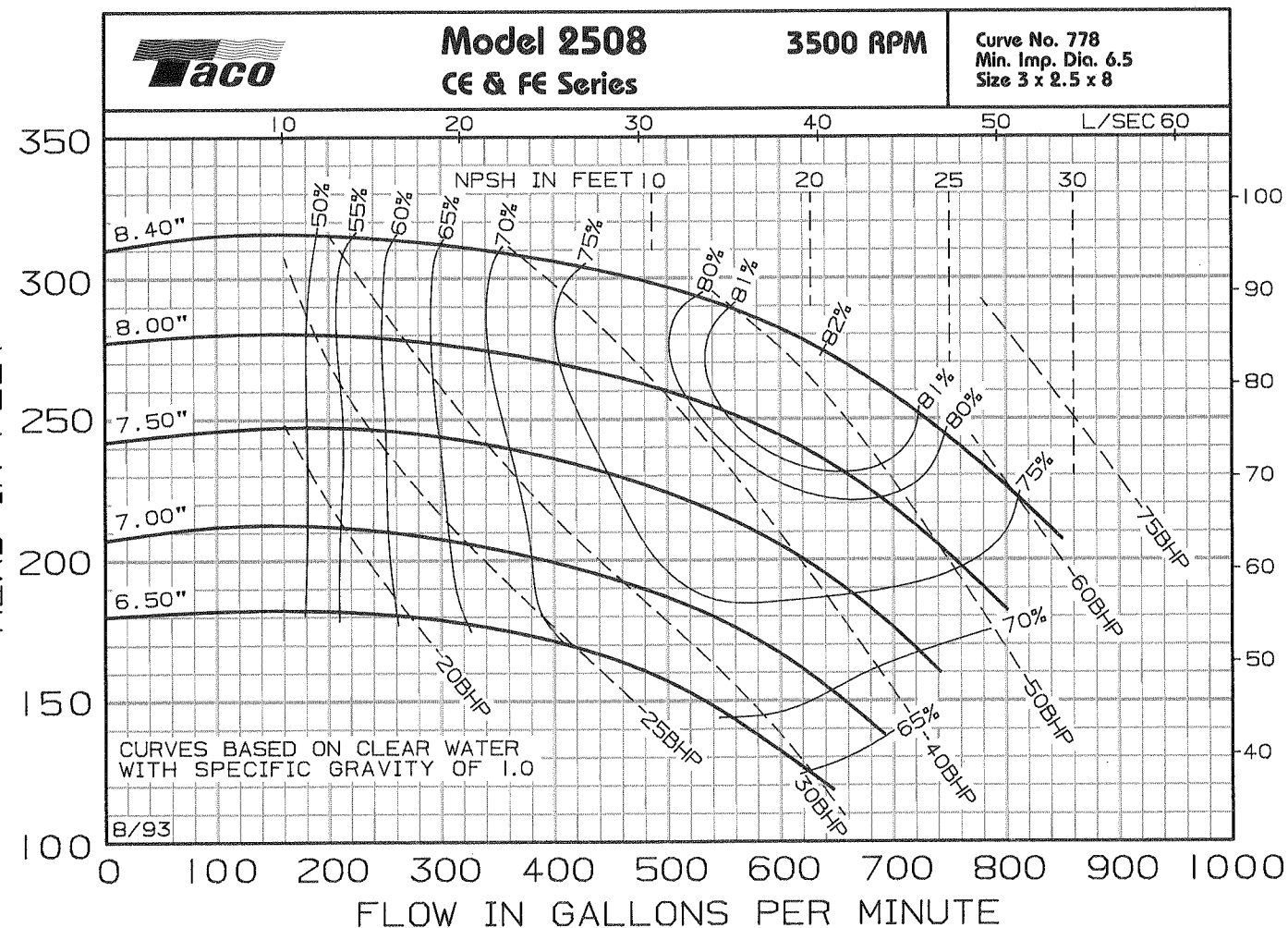
HEAD IN FEET



HEAD IN FEET



HEAD IN FEET



HEAD IN METERS



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